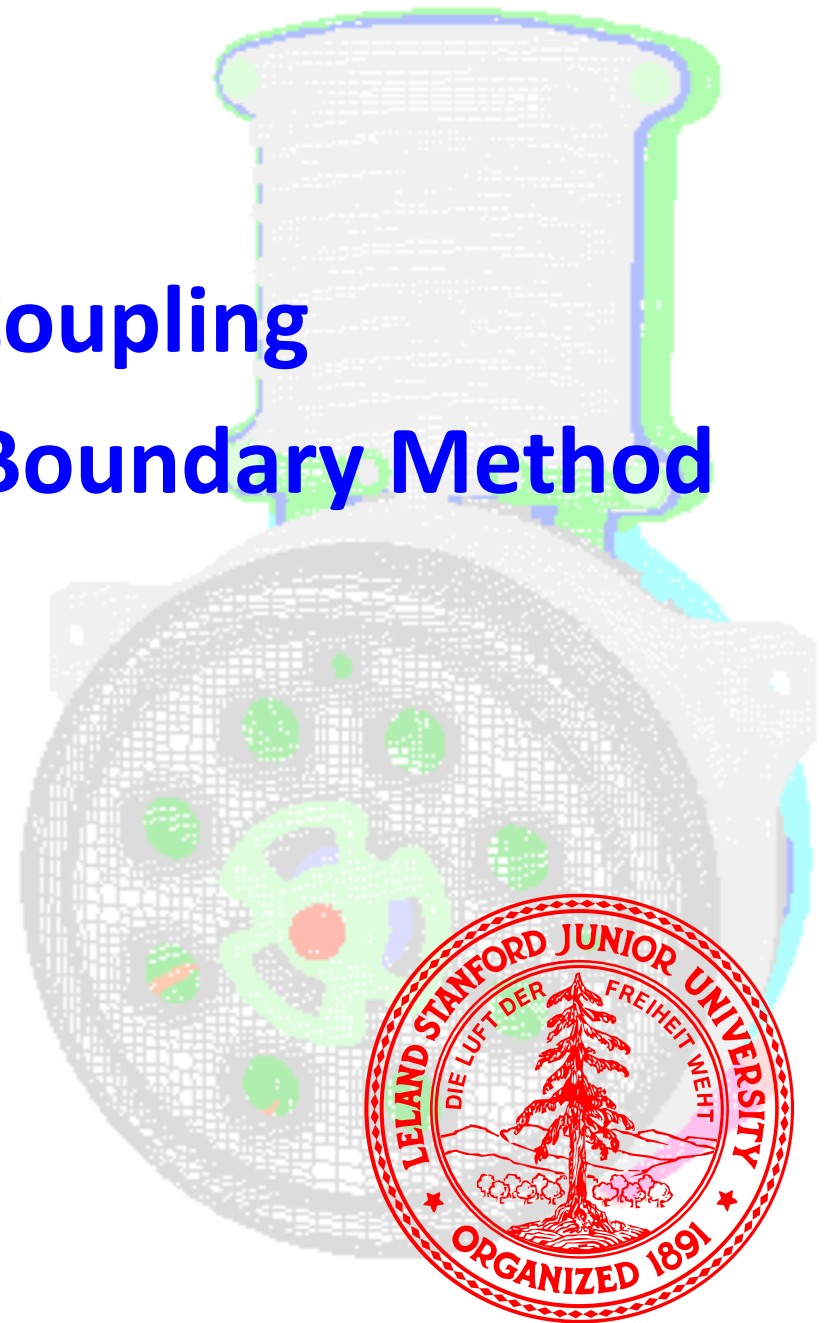


# Solid/Fluid Thermal Coupling Using the Immersed Boundary Method

Gianluca Iaccarino  
Mechanical Engineering  
Stanford University

XXXV WSC Conference  
October 6-10, 2010, Zeist, The Netherlands.

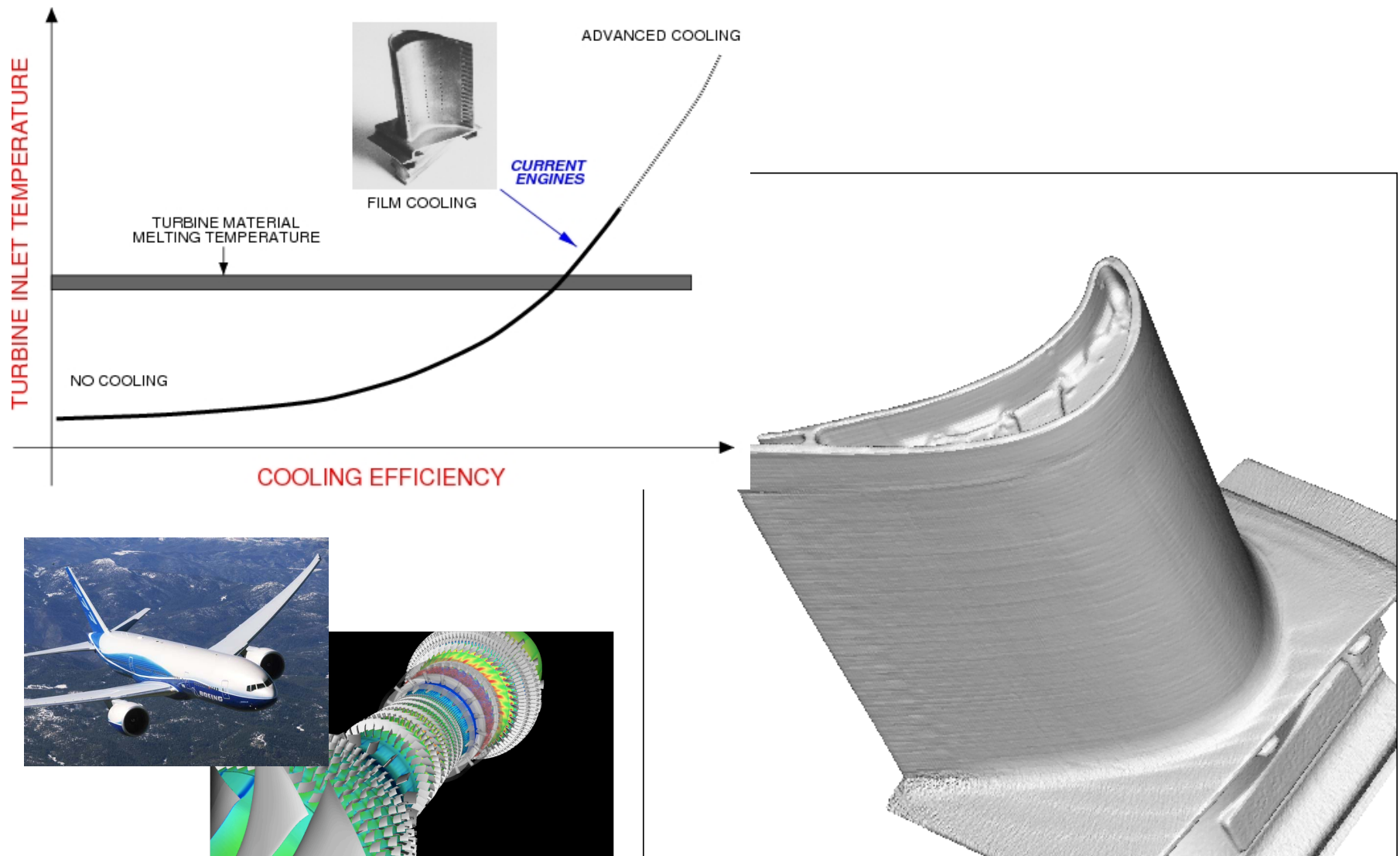


# **1. Motivations & Objectives**

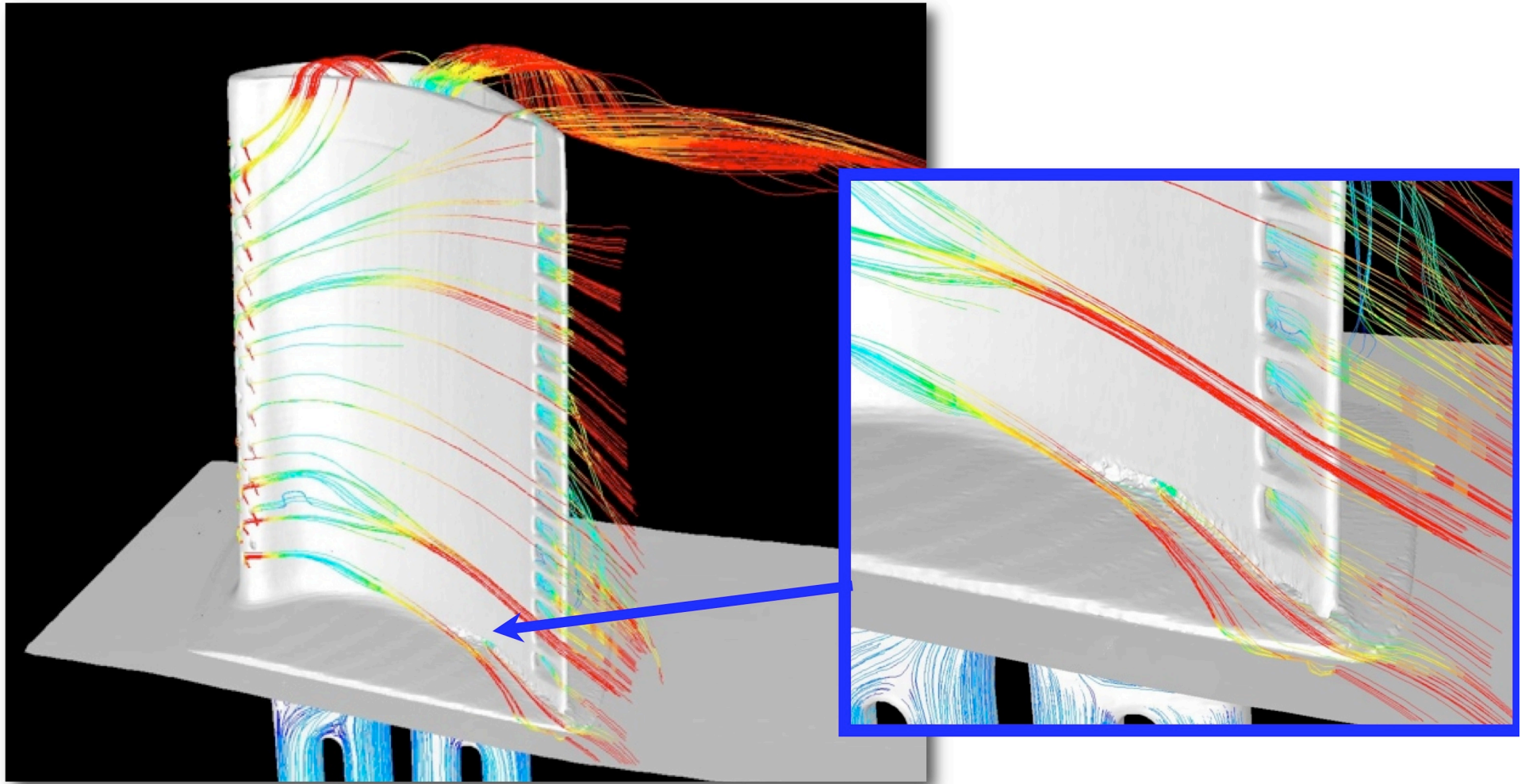
Solid/Fluid Thermal Coupling  
Using the Immersed Boundary Method

# **Why Solid/Fluid Thermal Coupling?**

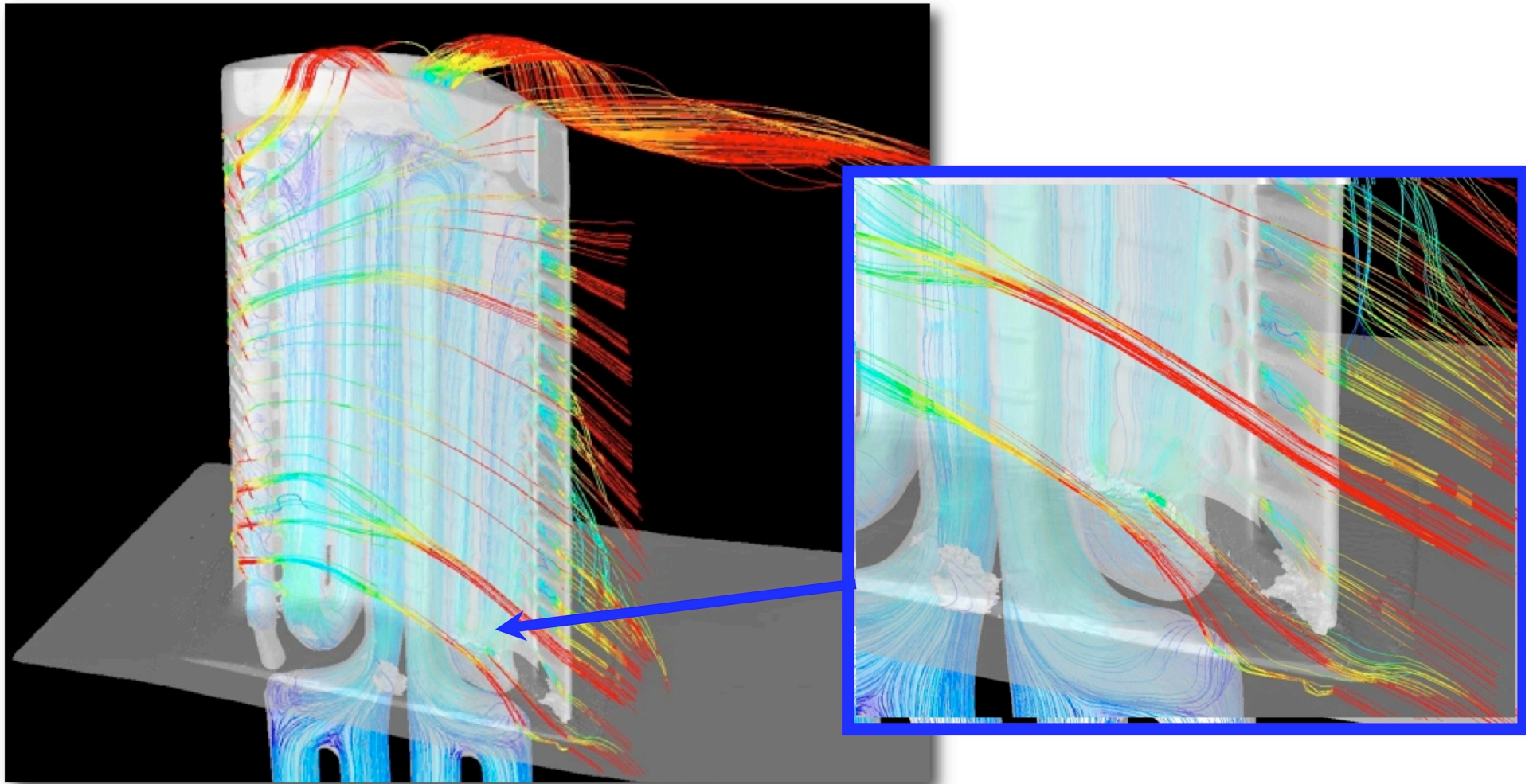
# Cooling of a Turbine Blade



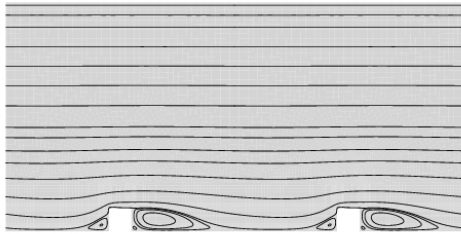
# Thermally Induced Damage?



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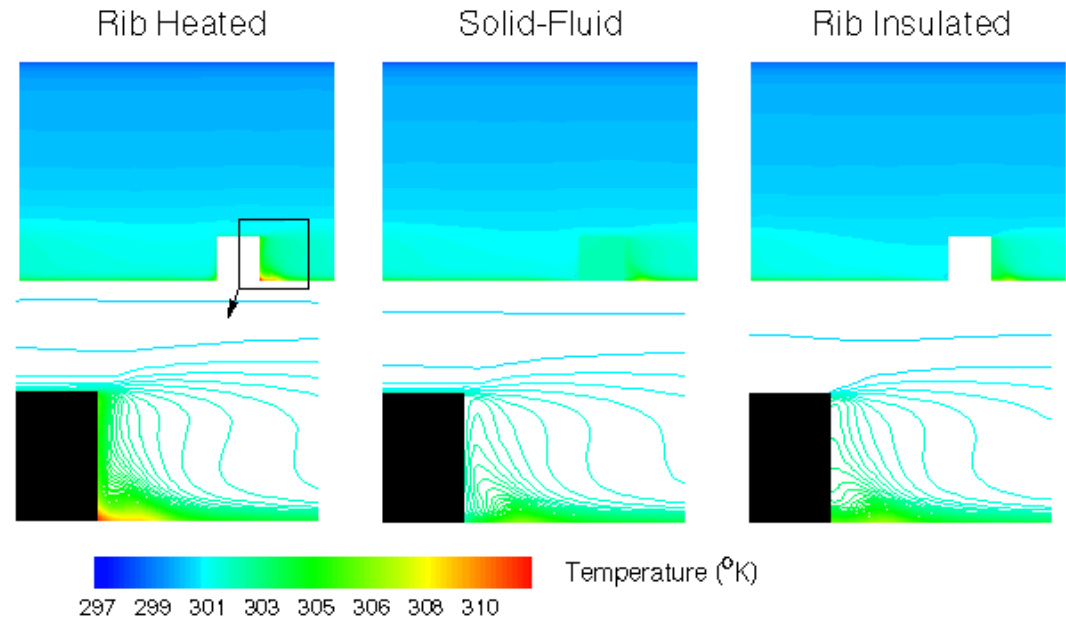
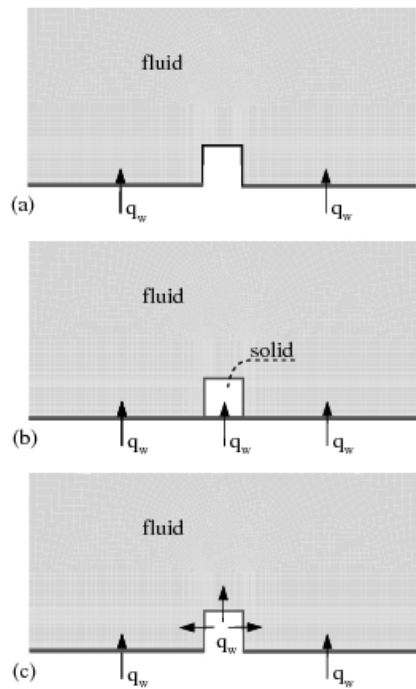


# Effect of Thermal B.C.

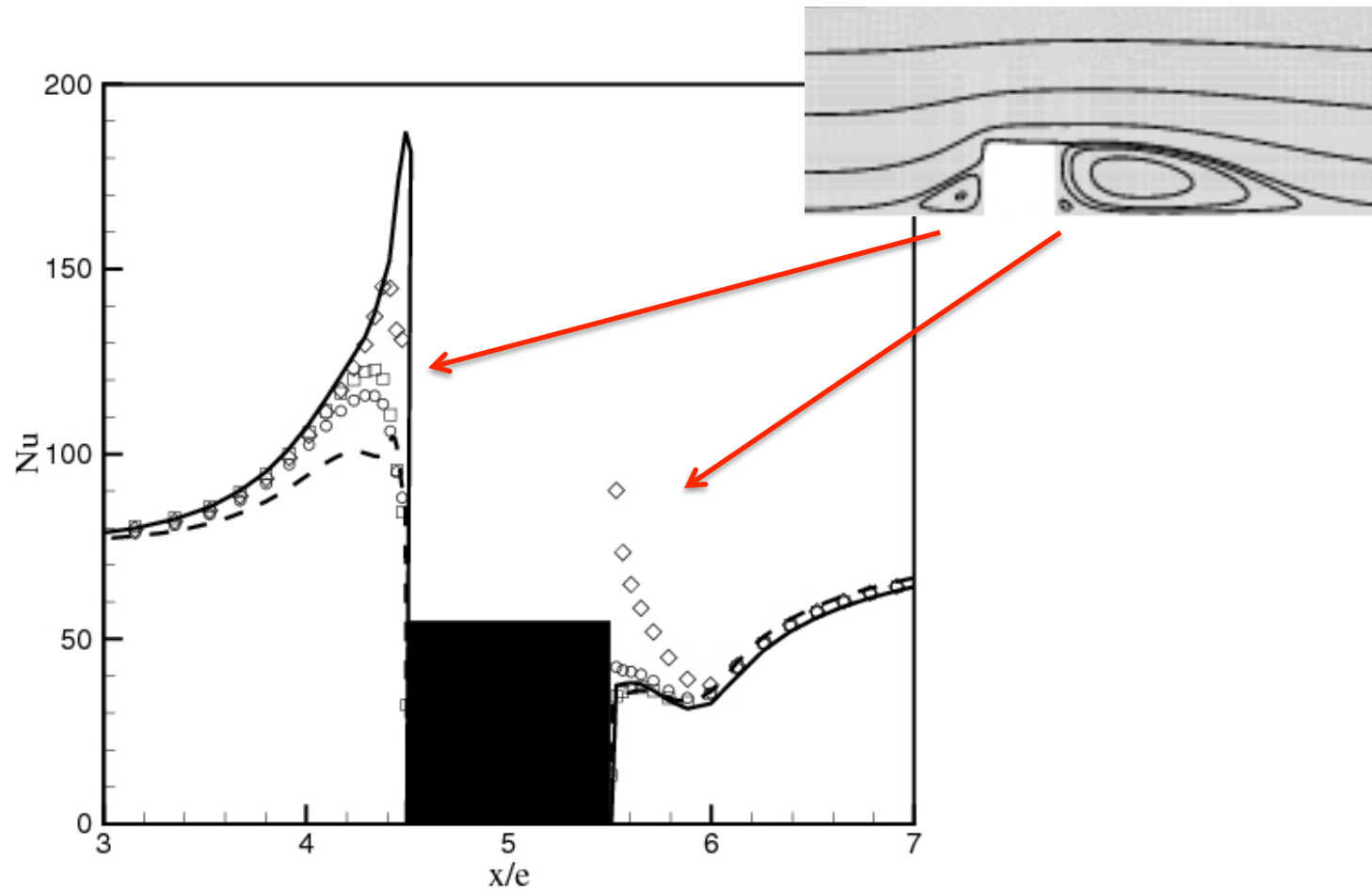


What is the appropriate thermal boundary condition?

Is it important to evaluate the wall heat transfer?



# Effect of Thermal B.C.



Fluid Only

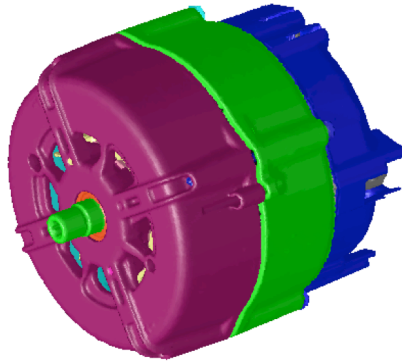
Solid/Fluid Coupled

— Insulated rib, --- heated rib,

◇  $k_s/k_f = 100$ , ○  $k_f = k_s$  and □  $k_f/k_s = 100$ .



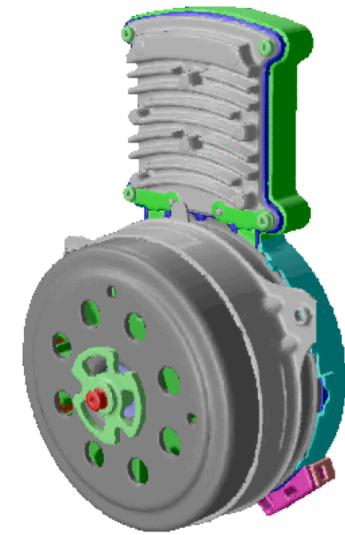
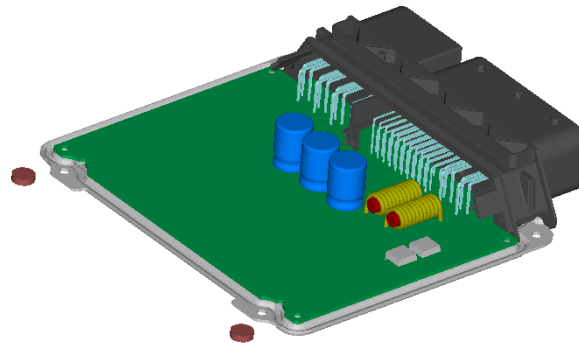
# Not only Turbine Blades



Alternator

**Valeo**

Electronic Component Unit (ECU)  **BOSCH**

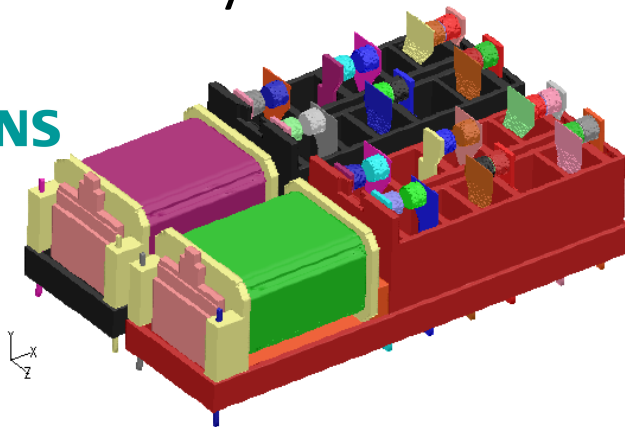


Electric Motor

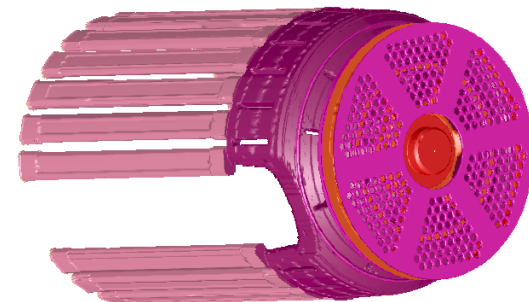
**Valeo**

Electronic Relay Board

**SIEMENS**



Electronic Motor  
**SIEMENS**



# The future is **GREEN**



Y2E2 Building @ Stanford

Natural Ventilation is a key  
for energy savings

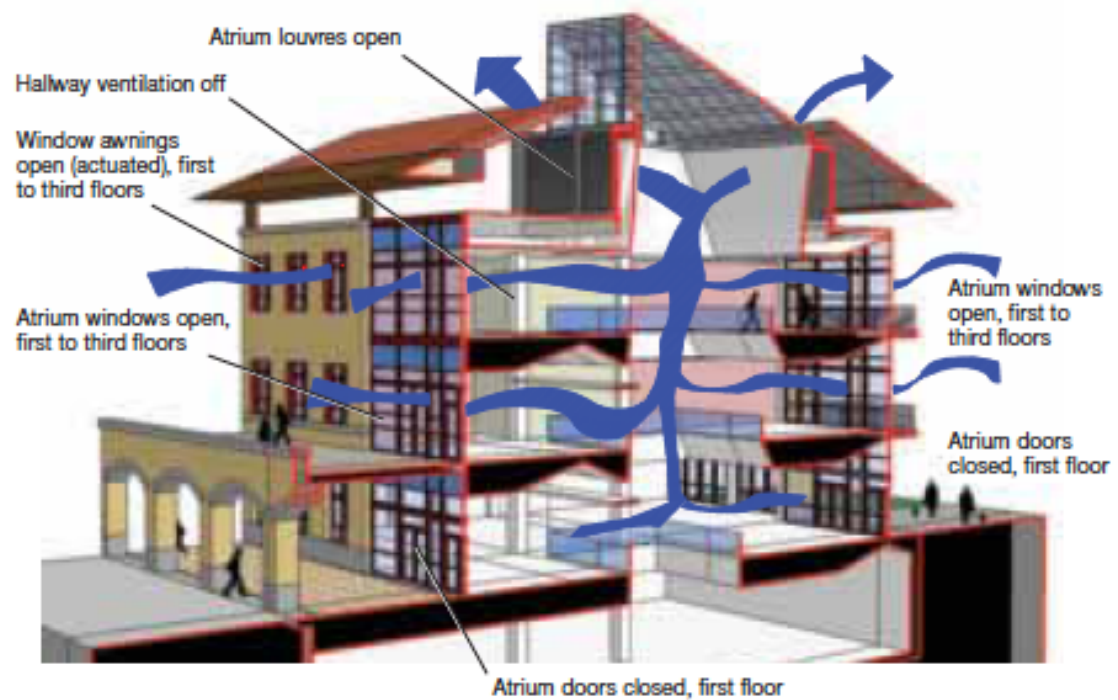


# The future is GREEN



Y2E2 Building @ Stanford

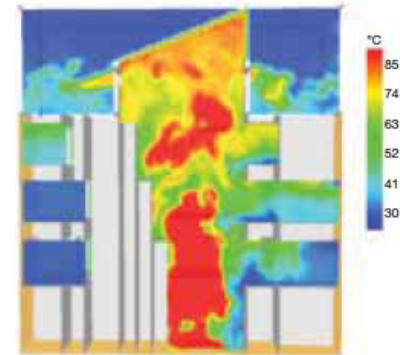
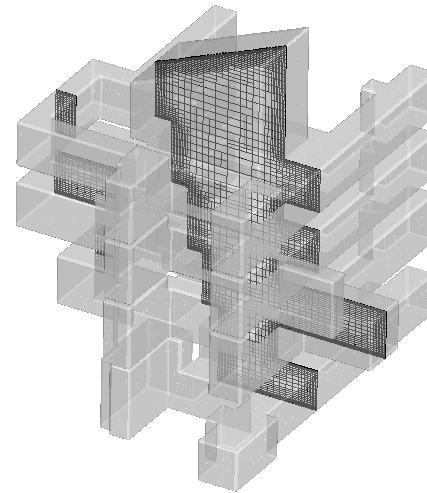
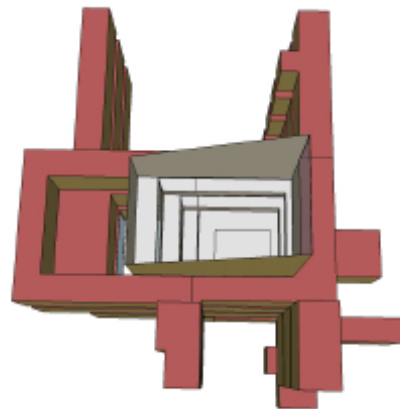
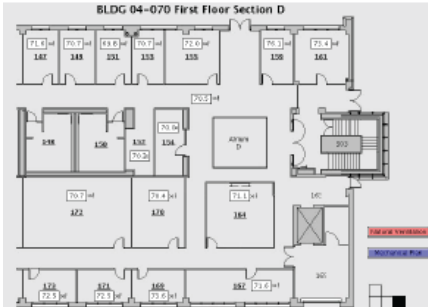
Objective: **simulate night-time air-flushing**



# The future is GREEN



Y2E2 Building @ Stanford



Enable CFD-aided Architectural Design

# Challenges

- CFD simulation of Conjugate Heat Transfer (CHT) are NOT routinely performed
  - **Geometry**: many parts, intricate passages, range of length scales
  - **Grid generation**: resolution requirement dependent on the part, time consuming
  - **Physical models**: treatment of multi-modal thermal transfer: natural/forced laminar/turbulent convection, conduction & radiation
  - **Coupling**: Enforce proper conservation at the solid/fluid interface

# Objective

- **Introduce** an Immersed Boundary method to perform coupled solid/fluid simulations in extremely complex geometries
- Provide a **technical description** and discuss **implementation details** of the approach
- Demonstrate the **accuracy** in simple applications and selected industrial problems

## **2. Immersed Boundary Methods**

Solid/Fluid Thermal Coupling  
Using the Immersed Boundary Method

# Immersed Boundary

Science is a differential equation, religion is the boundary condition

– A. Turing

The differential equation is the science, the boundary condition is a religion

– G. Iaccarino

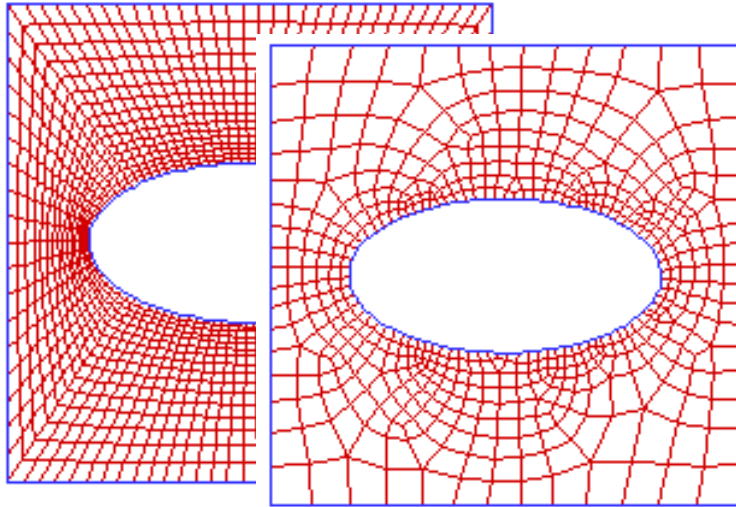
- A variety of research areas:
  - Develop numerical methods
  - Study fluid mechanics in complex domain (perhaps with moving boundaries)
  - Interested in interactions between fluid and structures
  - Analyze multiphase flows
  - ...



# Alternative Approaches

## Body-Fitted

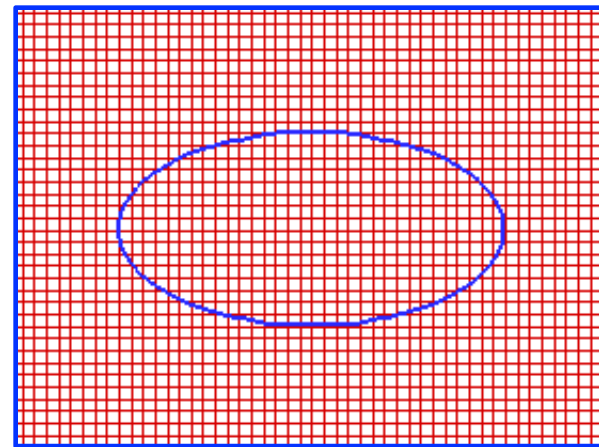
- The computational domain is contained within physical boundaries
- A **body-fitted mesh** is generated in the domain
- Solution algorithms handle structured or unstructured grids



Structured Grid or  
Unstructured Grid

## Immersed Boundary

- The computational domain extends beyond the physical boundaries
- A **Cartesian mesh** is generated in the domain
- The governing equations are **modified** in the cells cut by the interface



Cartesian Grid

# Scaling Argument

Desired resolution is  $\Delta n$ ,  $\Delta t$  at the boundary

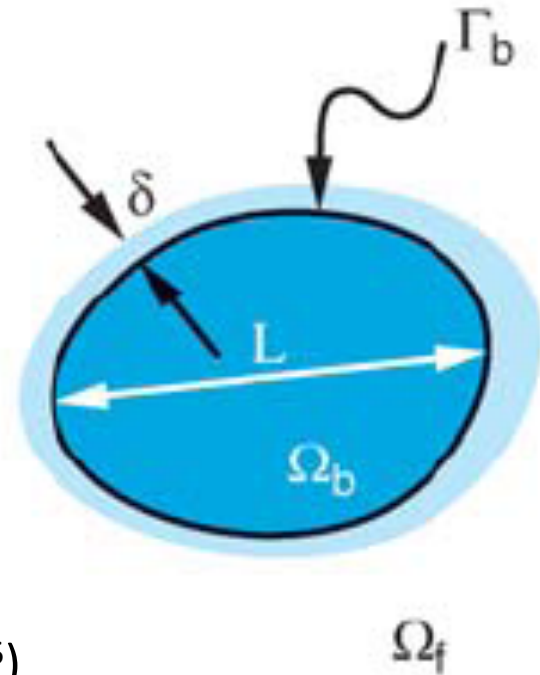
Assume  $\delta \ll L$

Body Conformal Grid  $\sim (L/\Delta t)(\delta/\Delta n)$

Cartesian Grid  $\sim (L/\Delta n)^2$

Assume  $\Delta n \sim \delta$  and  $\Delta t \sim L$  (and with  $L/\delta = Re^{0.5}$ )

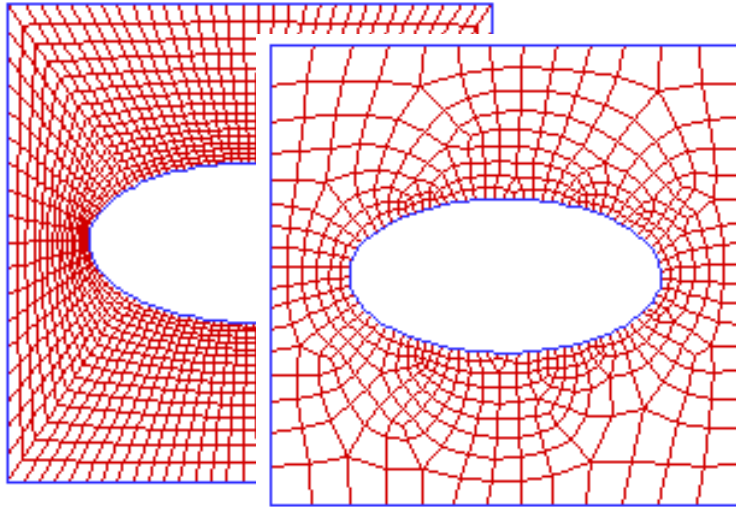
The grid-size ratio (in 2D) scales as the Re number: the IB is progressively more expensive....



# Alternative Approaches

## Body-Fitted

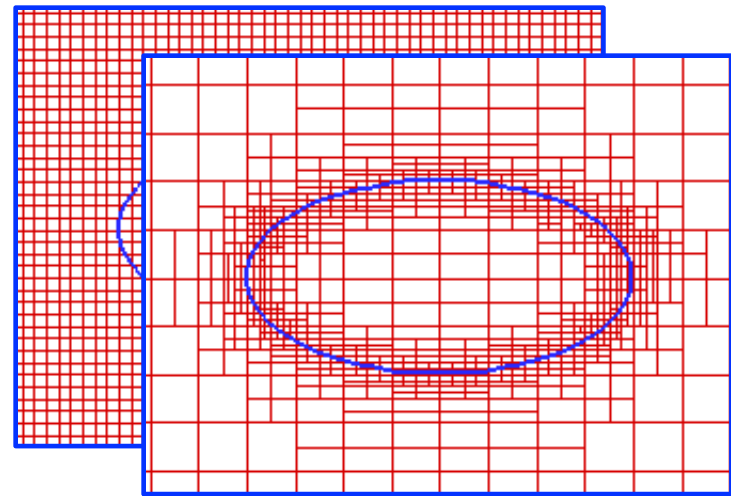
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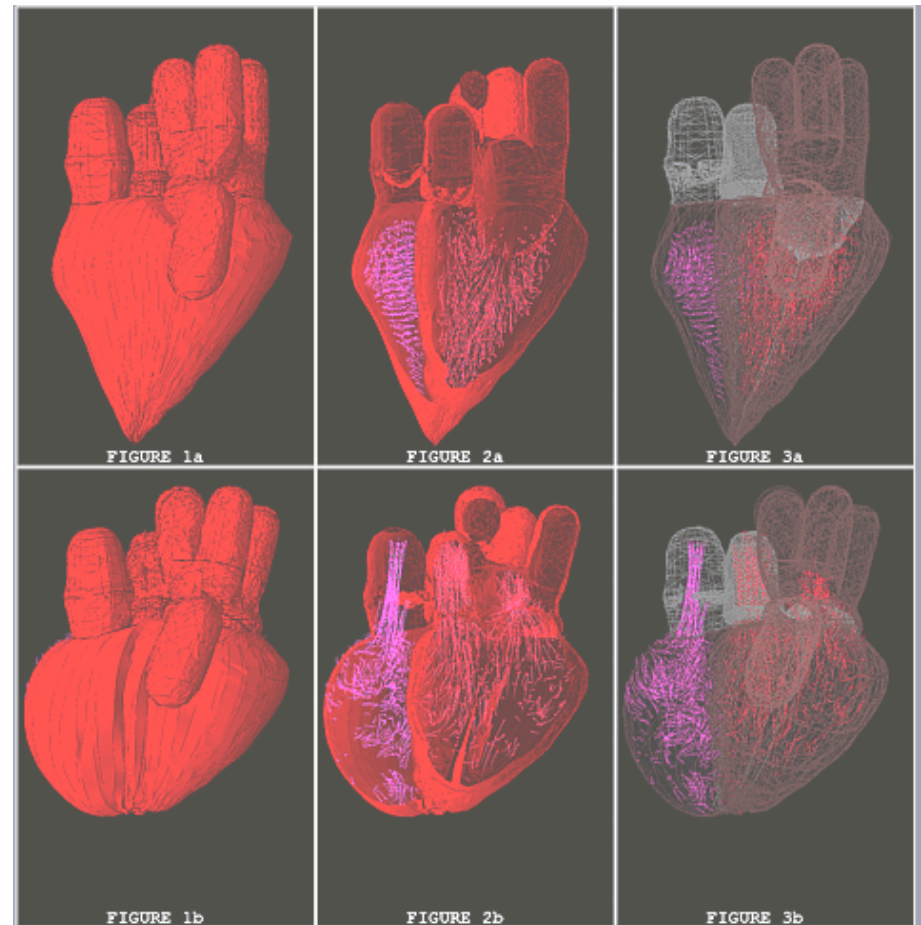


Cartesian Grid  
Unstructured Grid

# IB Methods: the origin

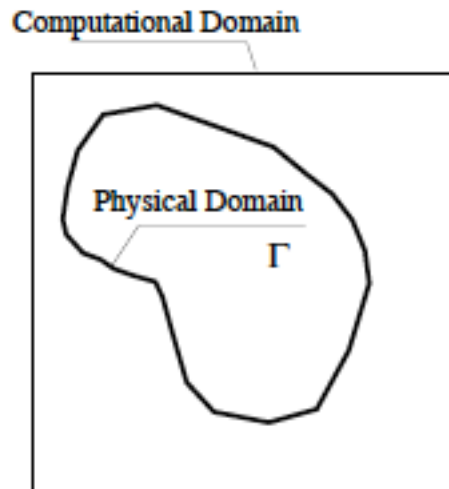
One father (many grandparents): Charles Peskin, 1972

1<sup>st</sup> ever simulation of a human heart-beat.



# IB Methods: the origin

1. The physical domain is the heart:  $\Gamma$   
The computational domain is a box.

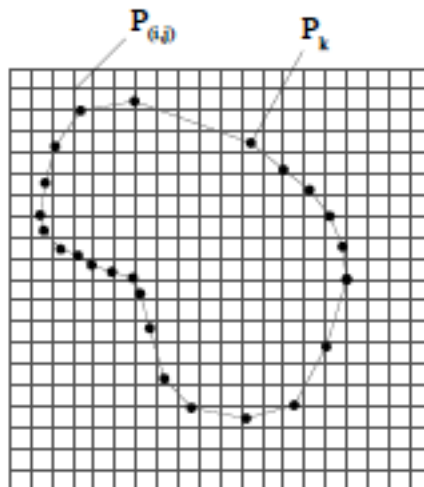
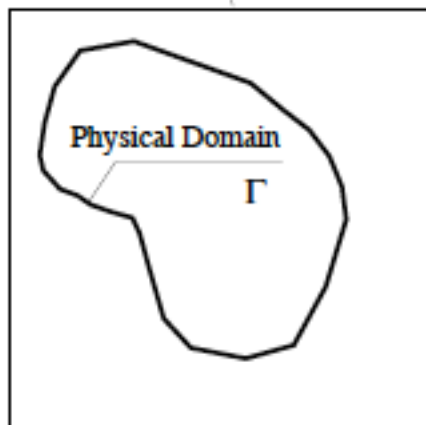


# IB Methods: the origin

2. A Cartesian, uniform grid covers the physical domain  
The Navier-Stokes Equations in Eulerian form are solved using a finite difference techniques

$$\frac{\partial \vec{u}}{\partial t} + \vec{u} \cdot \nabla \vec{u} + \frac{1}{\rho} \nabla p - \frac{\mu}{\rho} \nabla^2 \vec{u} = \vec{f}_m(\vec{x}, t)$$
$$\nabla \cdot \vec{u} = 0$$

Computational Domain



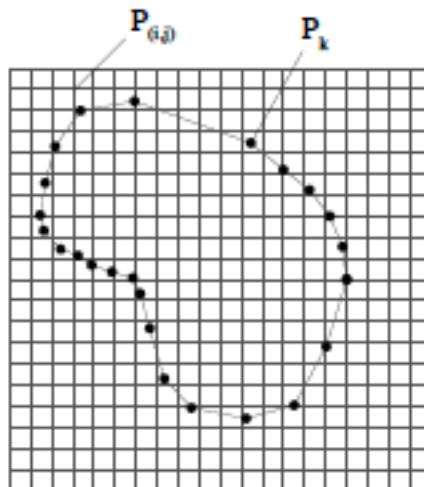
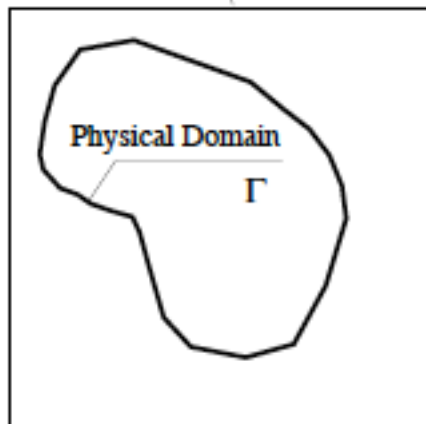
$\vec{f}_m(\vec{x}, t)$  is the force exerted by the heart on the fluid (at every location in space)

# IB Methods: the origin

3. A set of points describing the heart walls are advected using a Lagrangian method and the local fluid velocity

$$\frac{\partial \vec{X}_k}{\partial t} = \vec{u}(\vec{X}_k, t)$$

Computational Domain



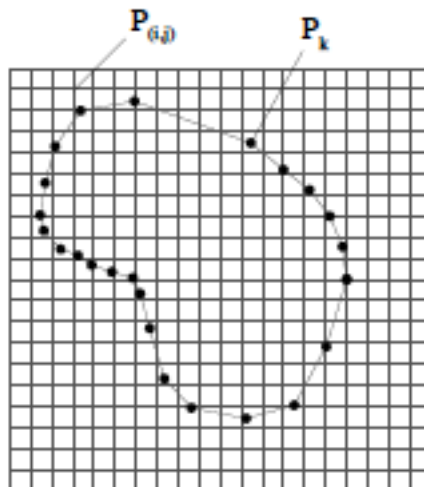
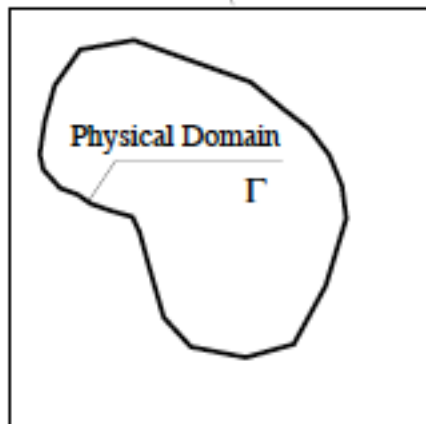
$\vec{u}(\vec{X}_k, t)$  is the velocity of the fluid at the location of the point  $k$

# IB Methods: the origin

4. Define the **coupling** between the heart and the fluid

$$\vec{f}_m(\vec{x}, t) = \sum_k \vec{F}_k(t) \delta(|\vec{x} - \vec{X}_k|)$$

Computational Domain



The coupling force depends on the model assumed for the hear wall:

- Points
- Fibers (Peskin)
- Membranes

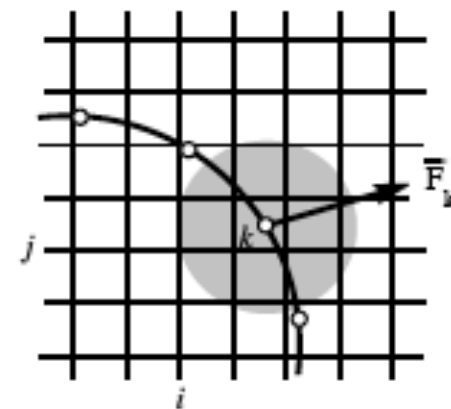
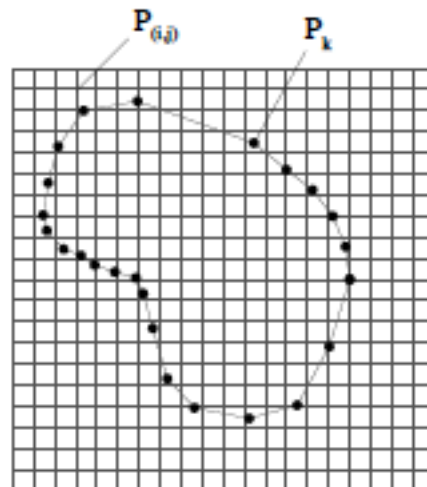
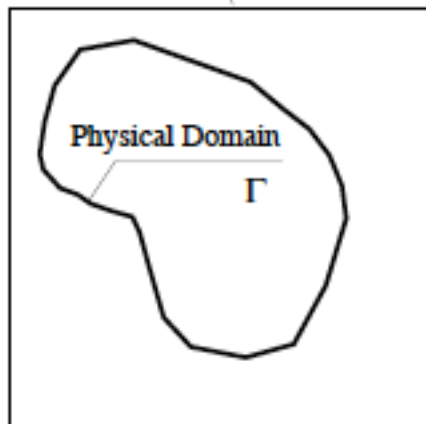


# IB Methods: the origin

5. **Transfer** the force from the heart to the fluid

$$\vec{f}_m(\vec{x}, t) = \sum_k \vec{F}_k(t) \delta(|\vec{x} - \vec{X}_k|)$$

Computational Domain



# IB Methods: the forcing function

$$\vec{f}_m(\vec{x}, t) = \sum_k \vec{F}_k(t) \delta(|\vec{x} - \vec{X}_k|)$$

In Peskin's approach it is derived directly from **Hooke's law**

For **rigid boundaries** various choices have been proposed:

$$\vec{F}_k(t) = -\kappa(\vec{X}_k - \vec{X}_k^e(t)) \quad \text{Peskin \& Lai (2000)}$$

$$\vec{F}_k(t) = (\mu/K)\vec{u}. \quad \text{Angot *et al.* (1998)}$$

$$\vec{F}_k(t) = \alpha \int_0^t \vec{u}(\tau) d\tau + \beta \vec{u}(t) \quad \text{Goldstein *et al.* (1993)}$$

# IB Methods: the forcing function

Peskin's IB approach is well suited for **elastic boundaries**

Standard forcing terms become ill-behaved in **the rigid limit**

**Ad-hoc forcing terms** (i.e. porosity) tend to be inaccurate and unstable

Definition of the forcing terms for turbulent quantities (as in RANS) is extremely challenging

Solution is required inside solid bodies

# IB Methods: the transfer function

$$\vec{f}_m(\vec{x}, t) = \sum_k \vec{F}_k(t) \delta(|\vec{x} - \vec{X}_k|)$$

Again various choices have been proposed:

$$\delta(r) \approx d_h(r) = \frac{1}{h} \begin{cases} (\cos(\pi r/2h) + 1)/4h, & \text{if } r \leq 2h, \\ 0 & \text{otherwise.} \end{cases}$$

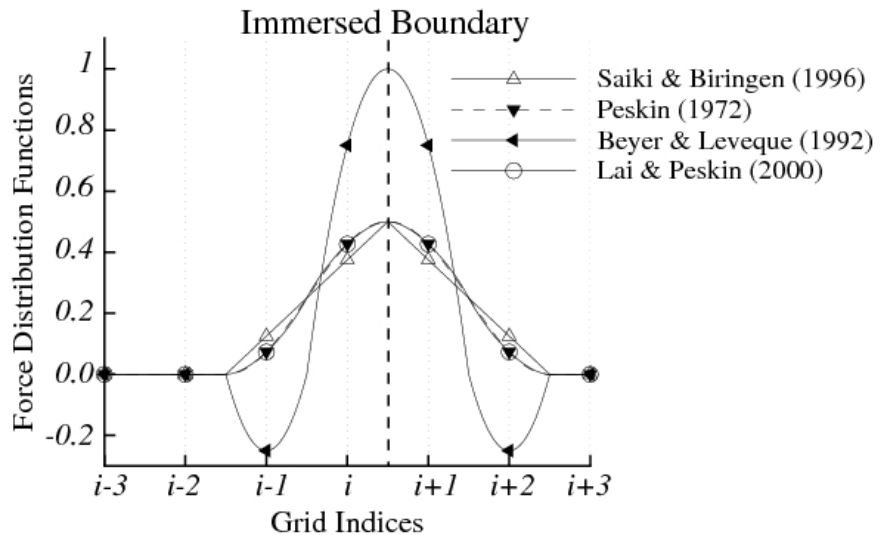
Peskin (1972)

$$\delta(r) \approx d_h(r) = \frac{1}{h} \begin{cases} (2h + r)/4h^2, & \text{if } r \leq 2h, \\ 0 & \text{otherwise.} \end{cases}$$

Peskin & Lai (2000)

$$\delta(r) \approx d_h(r) = \frac{1}{h} \begin{cases} 1 - (r/h)^2, & \text{if } r \leq h, \\ 2 - 3r/h + (r/h)^2, & h \leq r \leq 2h, \\ 0 & \text{otherwise.} \end{cases}$$

Beyer & Leveque (1992)



# IB Methods: the forcing function

The **transfer of the forcing functions** in Peskin's approach implies a non-sharp representation of the boundary which is not appropriate for boundary layers and heat transfer problems

Direct reference to the grid size  $h$  make the force not well suited for general locally refined grids

Some formulations result in forces that do NOT remain positive

**...but does this matter?**

# A simple example

Channel flow with a fixed horizontal membrane

Navier-Stokes Equations

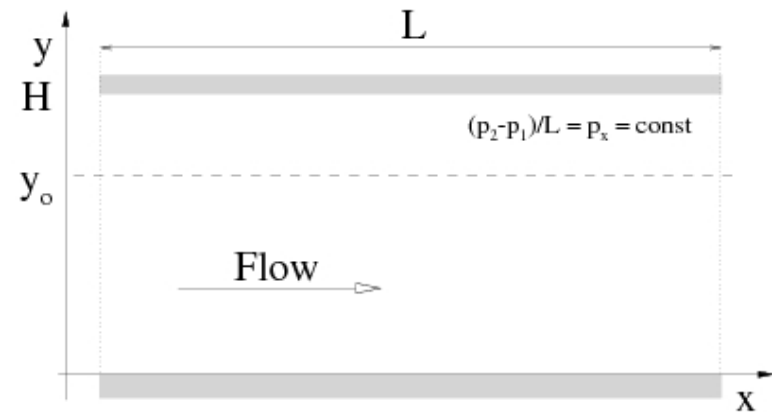
$$\frac{\mu}{\rho} \frac{d^2 U}{dy^2} = \frac{1}{\rho} p_x + F \delta(y - y_0)$$

Boundary Conditions

$$U(y = 0) = 0$$

$$U(y = H) = 0$$

$$U(y = y_0) = 0$$



# A simple example

Channel flow with a fixed horizontal membrane

Exact solution (2 parabolic flows)

$$U(y) = \begin{cases} (y/2\mu)(y - H)p_x - Fy(1 - y_o/H)(\rho/\mu) & \text{if } y \leq y_o, \\ (y/2\mu)(y - H)p_x - Fy_o(1 - y/H)(\rho/\mu) & \text{otherwise.} \end{cases}$$



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From the boundary conditions we can obtain:

$$F = -Hp_x/2\rho$$

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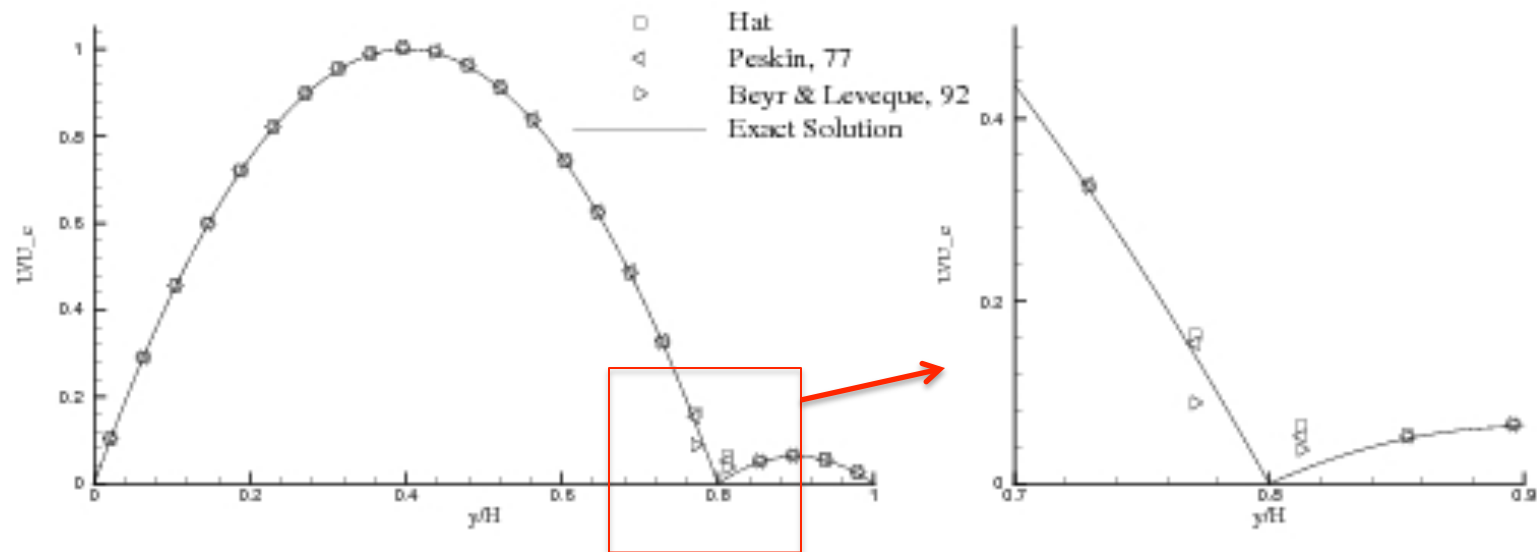
Note: velocity is continuous but not its derivative

$$\mu/\rho[dU/dy] = F.$$

# The effect of the force transfer

Channel flow with a fixed horizontal membrane

Use the exact force and various form of the transfer function (discrete delta function)



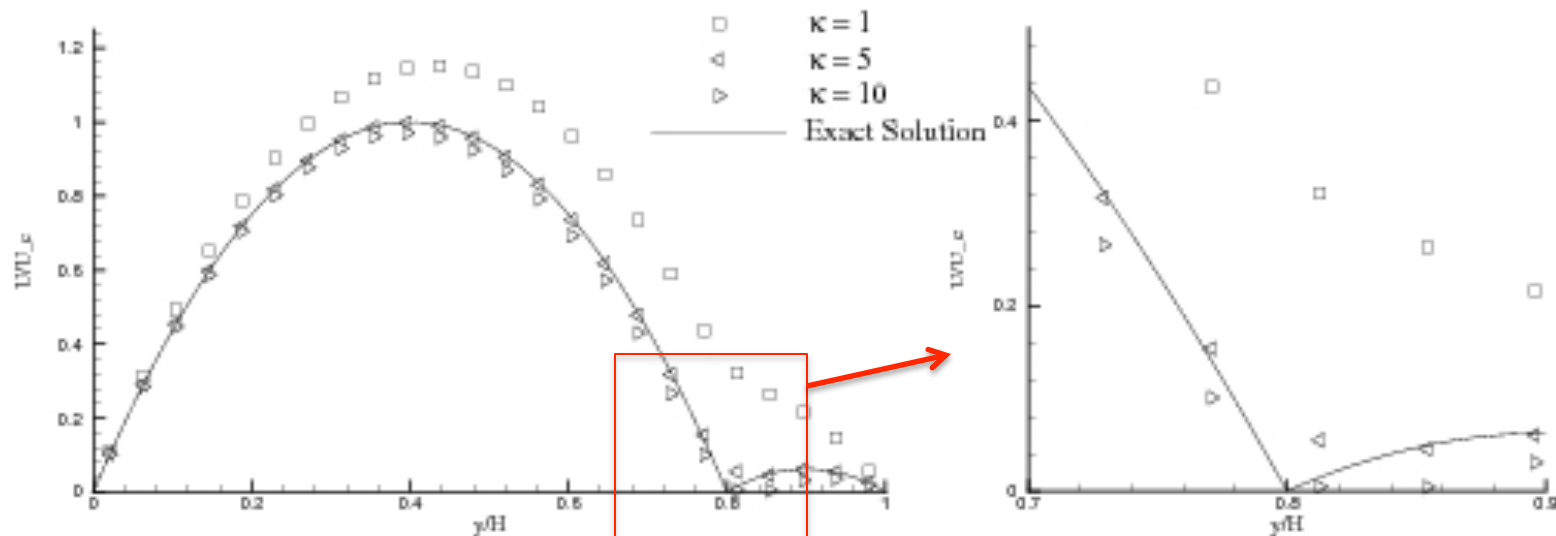
# The effect of the forcing

Channel flow with a fixed horizontal membrane

Approximate the force and use the BL transfer function

$$\vec{F}_k(t) = (\mu/K)\vec{u}.$$

Angot *et al.* (1998)

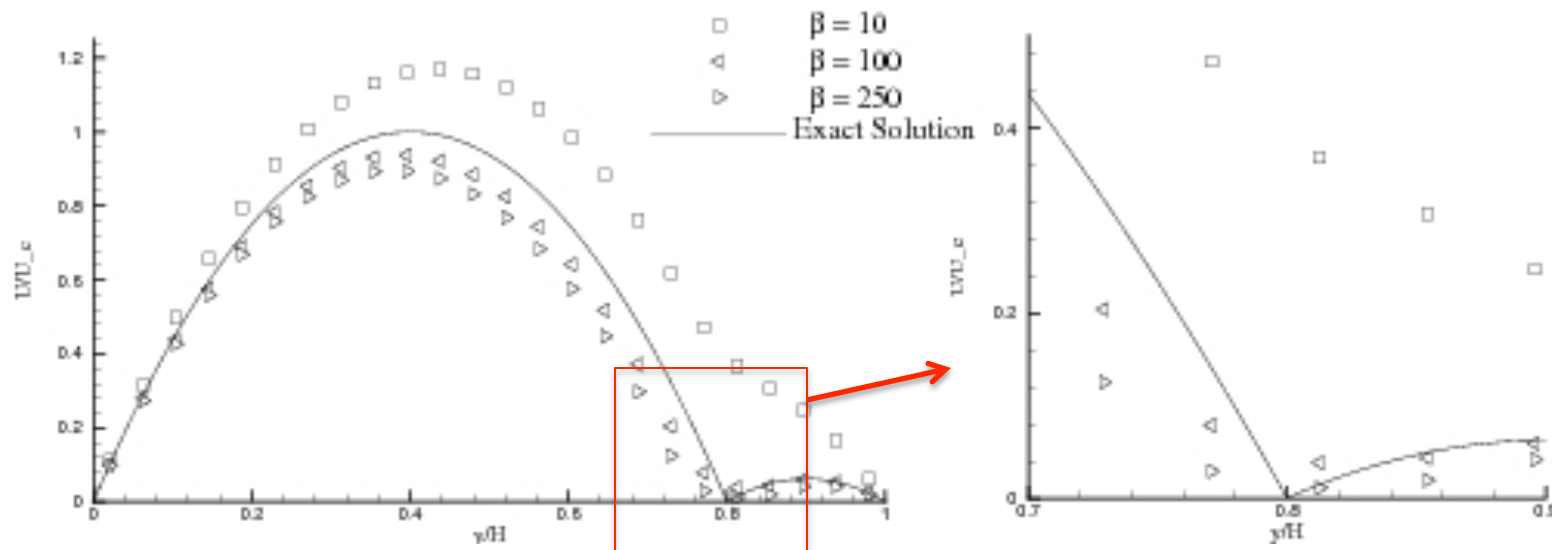


# The effect of the forcing

Channel flow with a fixed horizontal membrane

Approximate the force and use the BL transfer function

$$\vec{F}_k(t) = \alpha \int_0^t \vec{u}(\tau) d\tau + \beta \vec{u}(t) \quad \text{Goldstein *et al.* (1993)}$$



# Summary

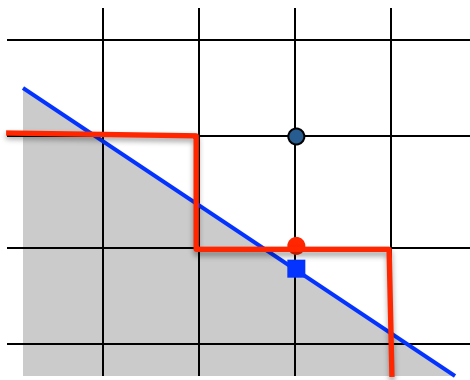
Both the choice of the **forcing function** and the **solid/fluid transfer** are important

In general it is difficult to distinguish between the errors introduced by each step

An additional difficulty is to ensure that **conservation properties** are preserved at the IB (mass conservation, etc.)

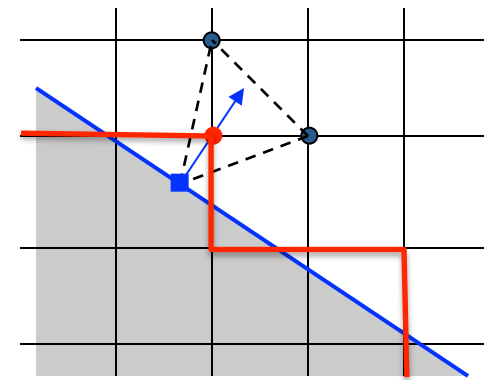
# An alternative IB method

Instead of representing the fluid/solid coupling as a continuous force, we **reconstruct** a “new” **virtual boundary condition** on a **modified domain**



1d reconstruction

- Fluid point (NS solution)
- True Boundary
- Virtual Boundary

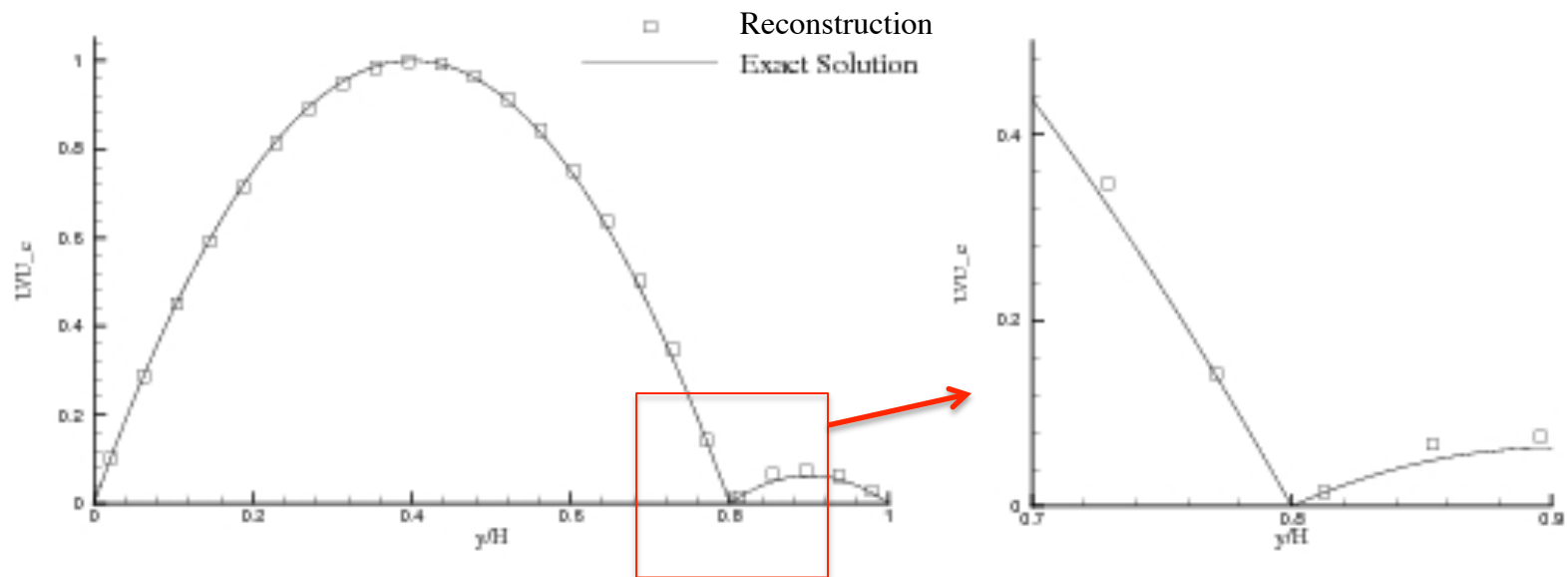


Multi-d reconstruction

# The effect of the reconstruction

Channel flow with a fixed horizontal membrane

1D Linear Reconstruction      Fadlun *et al.* (2000)





# Reconstruction vs. Forcing

Reconstruction is applied discretely, therefore can be “synchronized” with the discretization scheme (**need to prove**)

Physical constraints can be added to the reconstruction, e.g. mass conservation, turbulent wall functions, etc. (**need to prove**)

Reconstructions are local and do not require uniform meshes and solid walls (or moving walls) do not require any special treatment (**need to prove**)

# **3. Geometry and Grid Generation**

Solid/Fluid Thermal Coupling

Using the Immersed Boundary Method

# Components

Description of the True Boundary

Definition of the Virtual Boundary

Grid Refinement Criteria

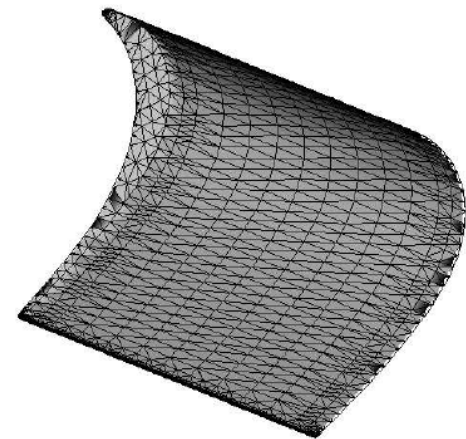
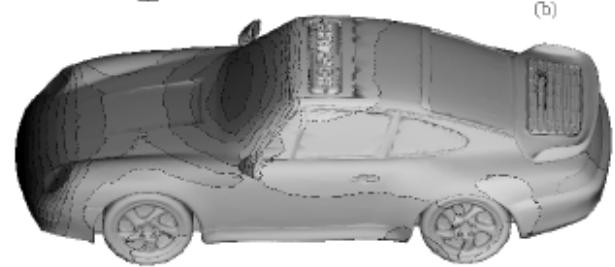
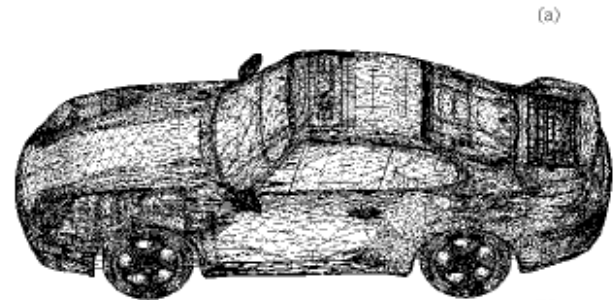
Handling Imperfect CAD Parts (digression – if time permits)

# Description of the True Boundary

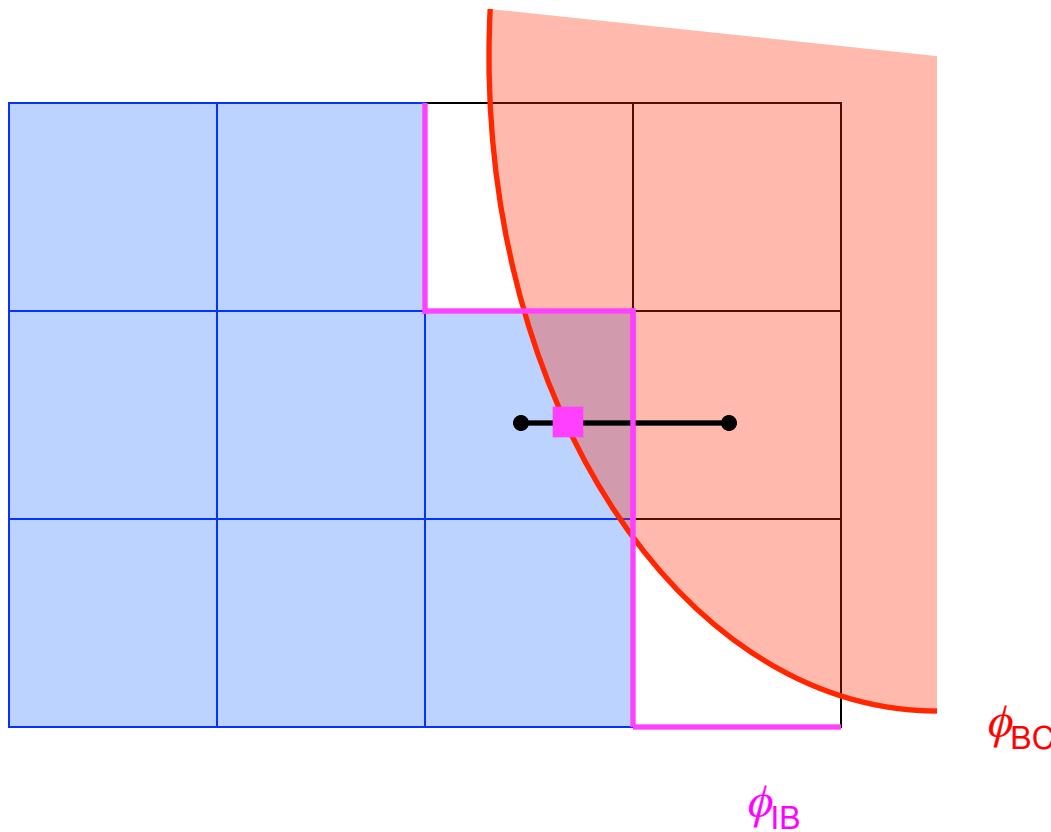
## Stereolithography Surfaces (STL)

### Advantages

- A set of triangles
- No high-order topological info
- Imperfections are tolerated (intersections, overlaps, etc.)



# Description of the Virtual Boundary

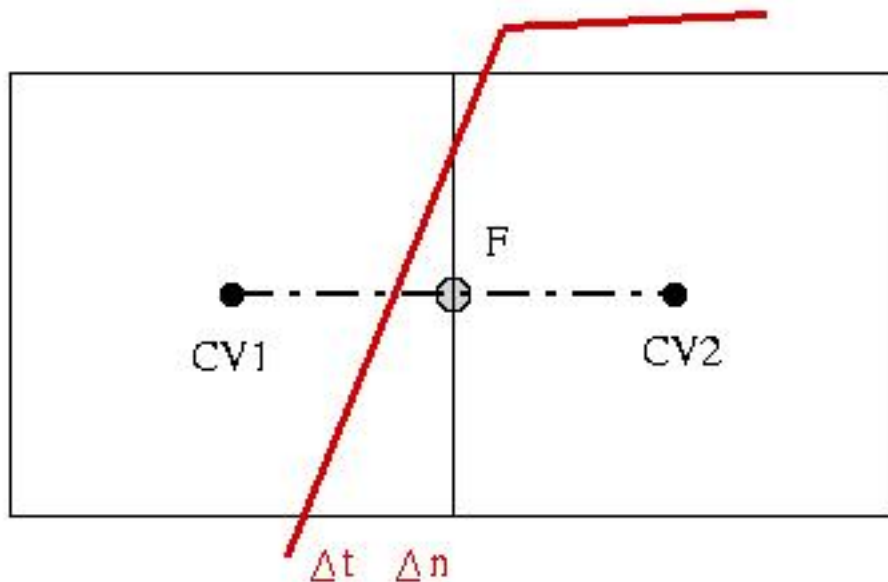


The relation between  $\phi_{BC}$  and  $\phi_{IB}$  involves geometrical quantities (e.g. surface normal) and physical constraints (e.g. wall model, conservation laws)

# Grid Refinement Criteria

The underlying meshes are locally refined – unstructured

Given the desired resolution at the STL surface ( $\Delta n$ ,  $\Delta t$ ) the intersections are used to **detect the CVs to refine**

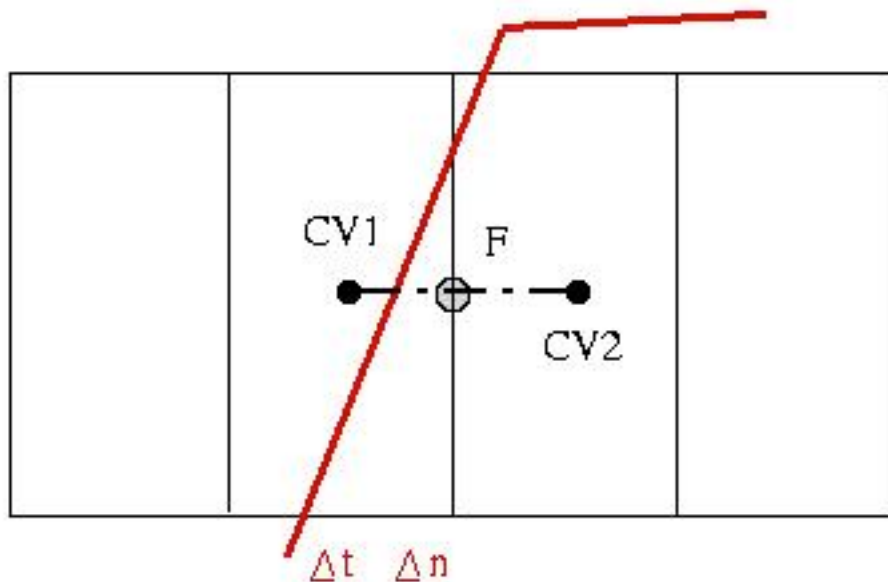


$$\Delta x_i^{CV} = \text{MIN} \left( \frac{\Delta n}{|n_i^{STL}|_i}; \Delta t \right)$$

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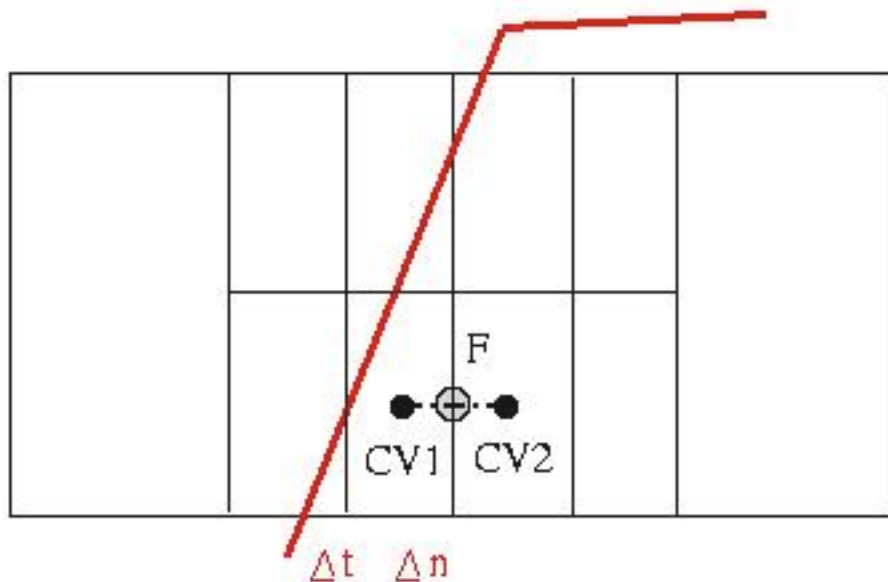


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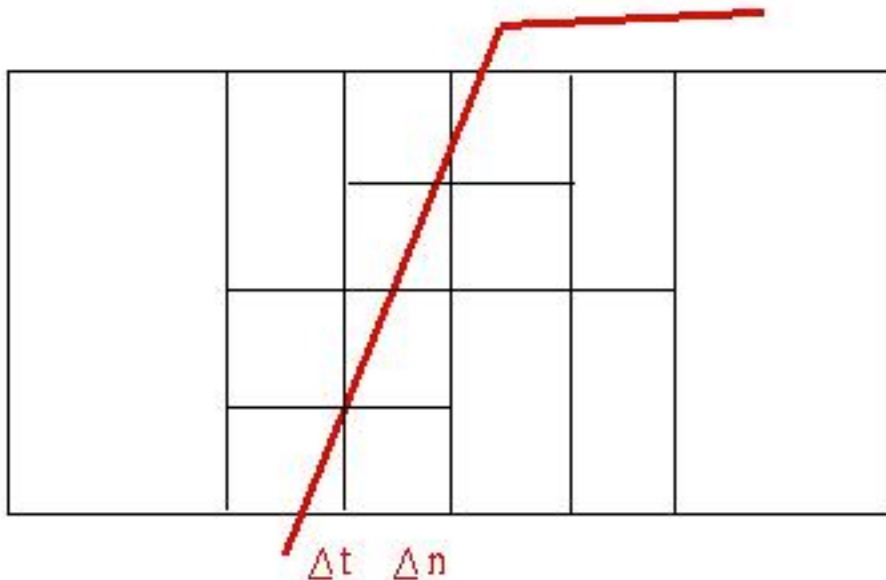
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The underlying meshes are locally refined – unstructured

Given the desired resolution at the STL surface ( $\Delta n$ ,  $\Delta t$ ) the intersections are used to **detect the CVs to refine**

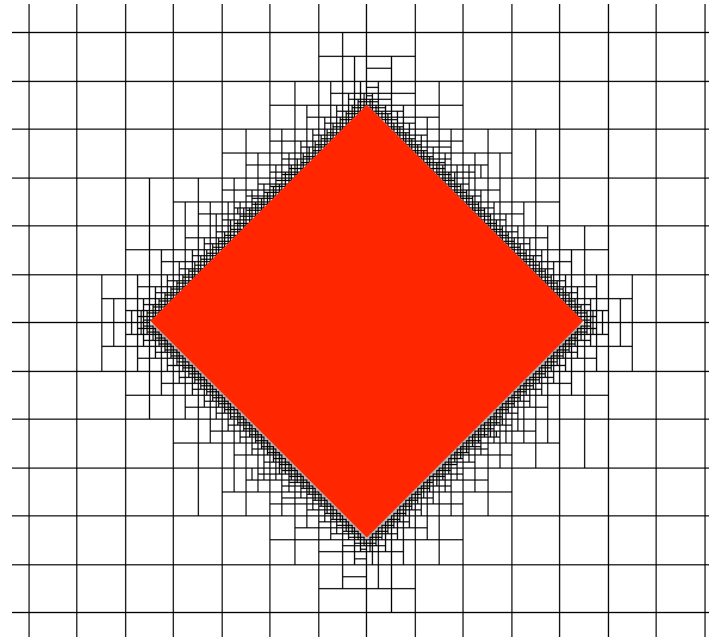
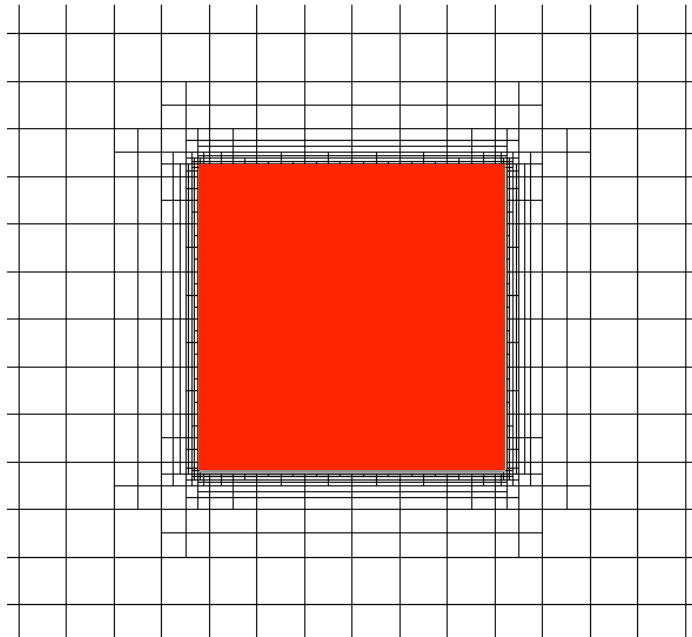


$$\Delta x_i^{CV} = \text{MIN} \left( \frac{\Delta n}{|n_i^{STL}|}; \Delta t \right)$$

# Grid Refinement Criteria

The underlying meshes are locally refined – unstructured

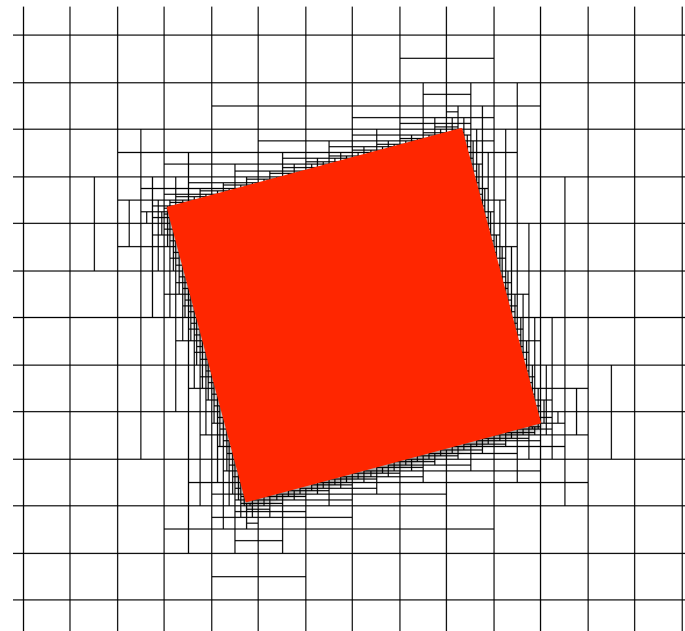
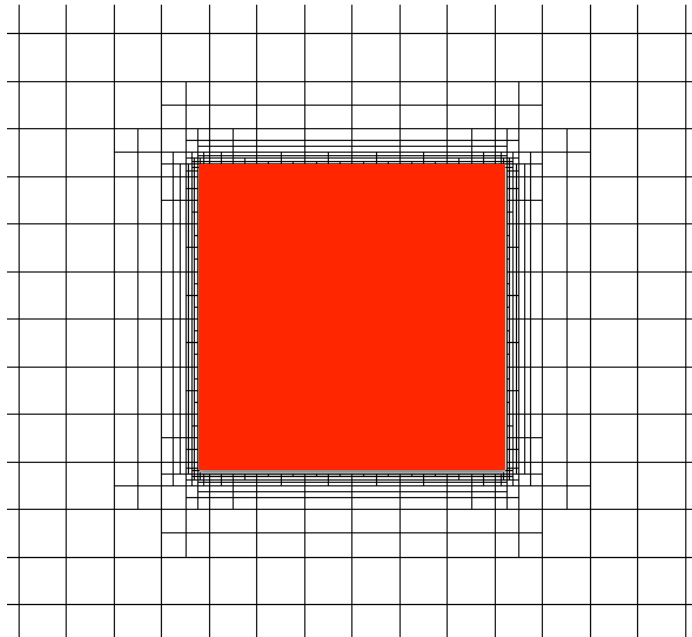
( $\Delta n$ ,  $\Delta t$ ) anisotropy results in reduced grid size for grid aligned STLs



# Grid Refinement Criteria

The underlying meshes are locally refined – unstructured

$(\Delta n, \Delta t)$  anisotropy results in reduced grid size for grid aligned STLs

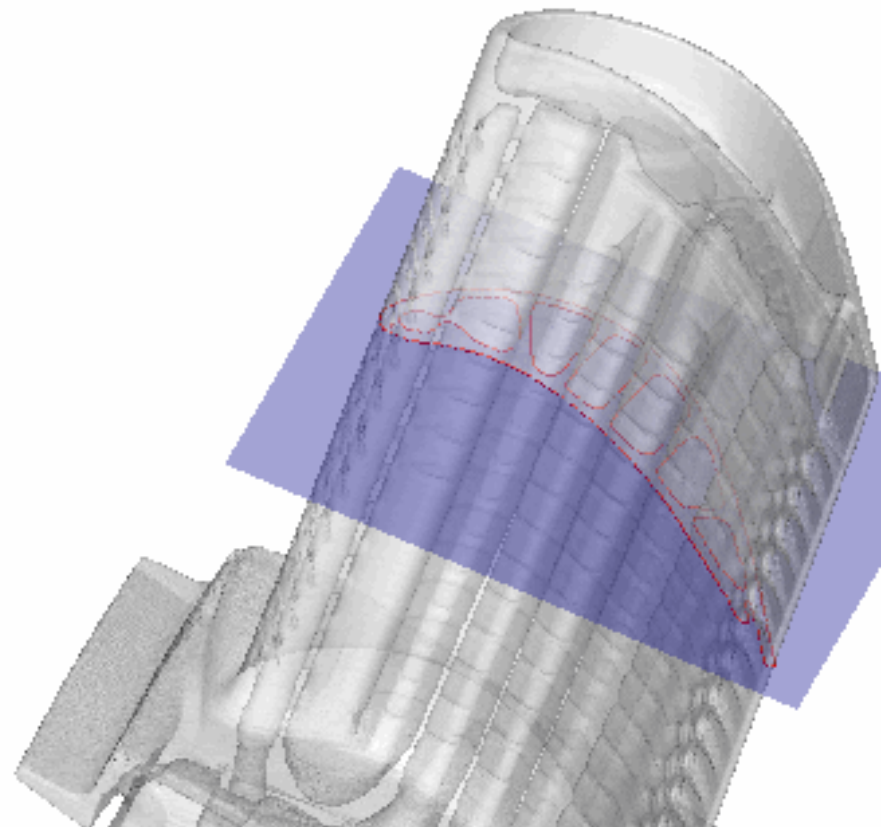
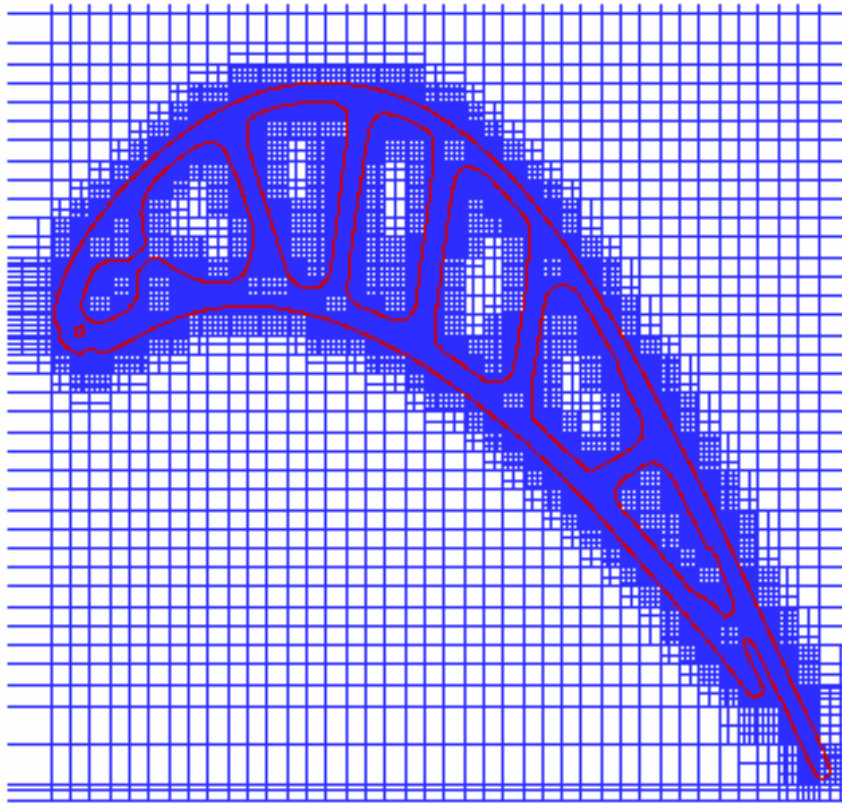


# A grid generation example

## Turbine Blade

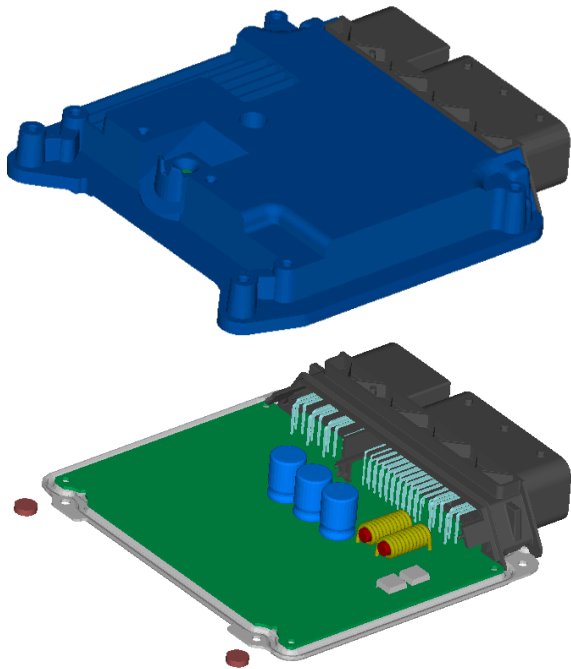
Horizontal Grid Cuts

Grid Generated using 4 Iterations of Ray Tracing



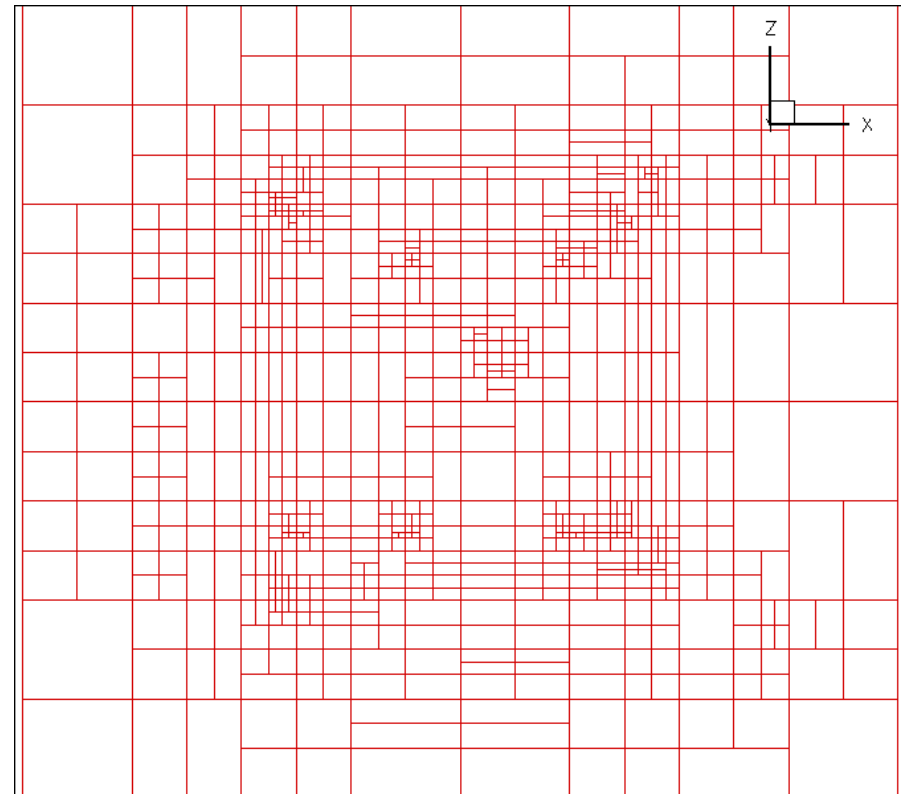
# A grid generation example

## Electronic Component Unit



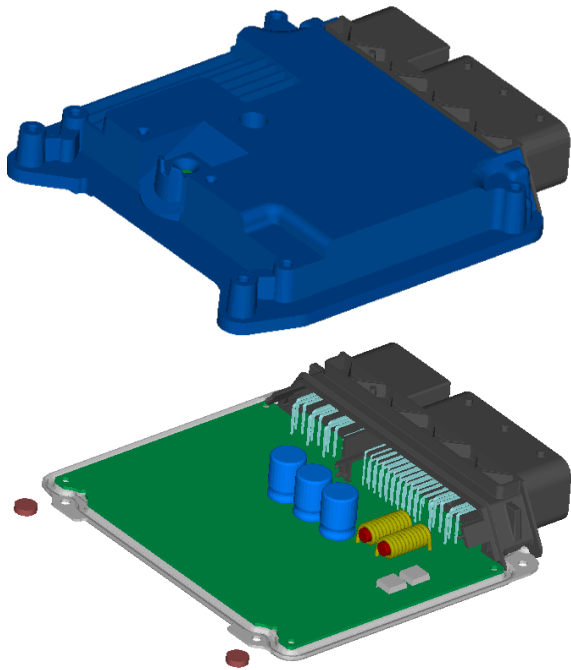
Horizontal Grid Cuts

Grid Generated using 6 Iterations of Ray Tracing

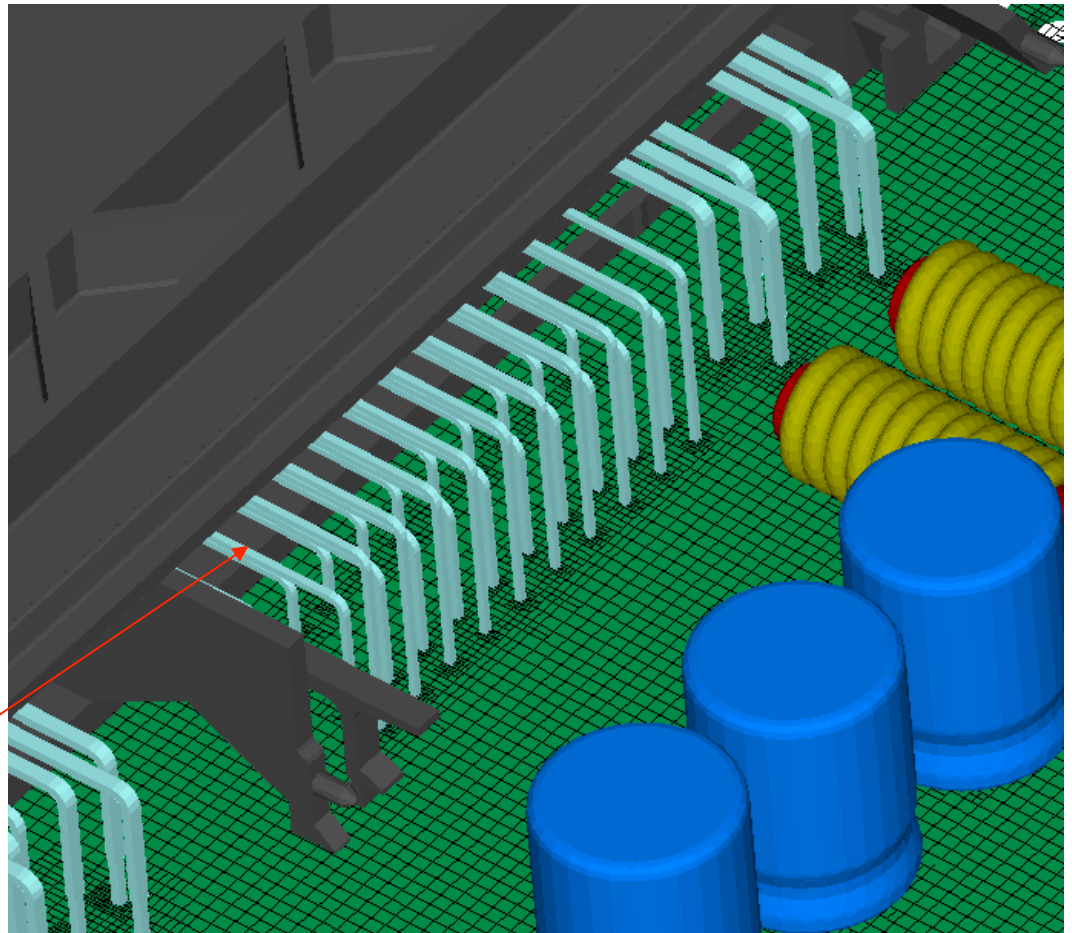


# A grid generation example

## Electronic Component Unit



Every single  
pin is captured



# Summary

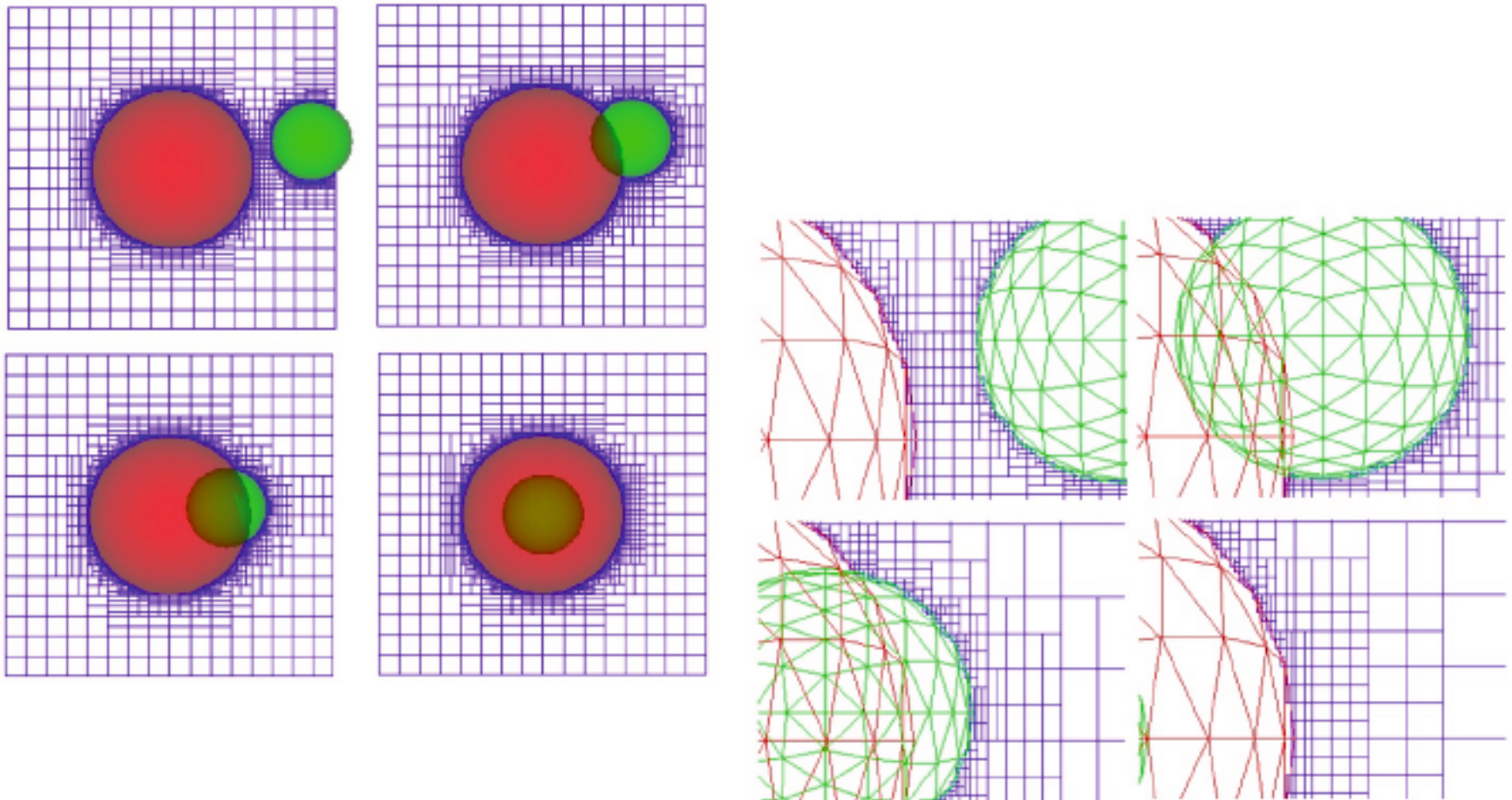
Combination of STL surfaces (triangles) and **ray tracing** provides a flexible infrastructure to generate the virtual boundary and the underlying grid

Unstructured grids with **local grid refinement** are efficient in achieving desired tangential and normal resolution

Unstructured environment is flexible in creating **cylindrical**, Cartesian or general curvilinear grids

# Handling Imperfect CAD Parts

Ray tracing is applied locally at each face (CV-CV segment), therefore it provides ample flexibility

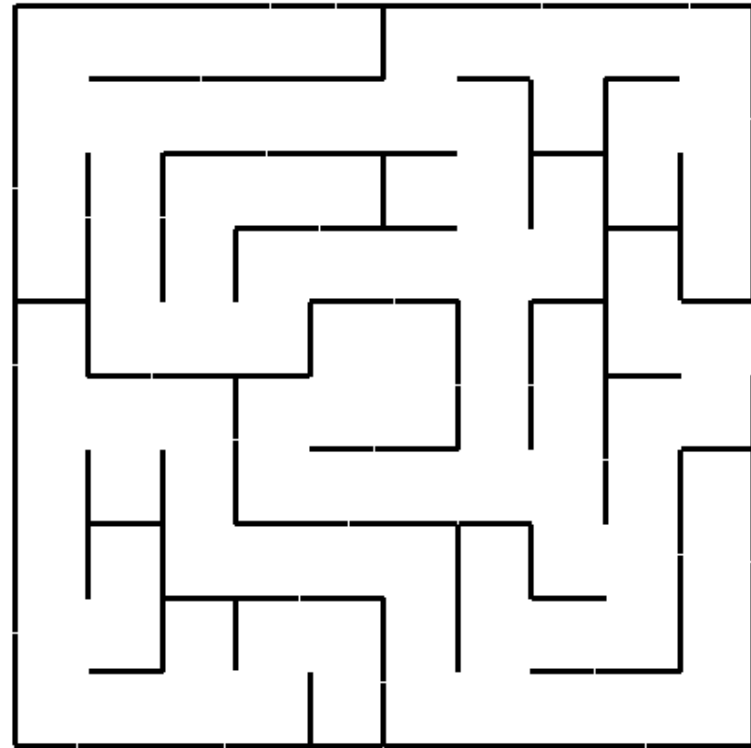




**(short) digression...**

# Find Leakages

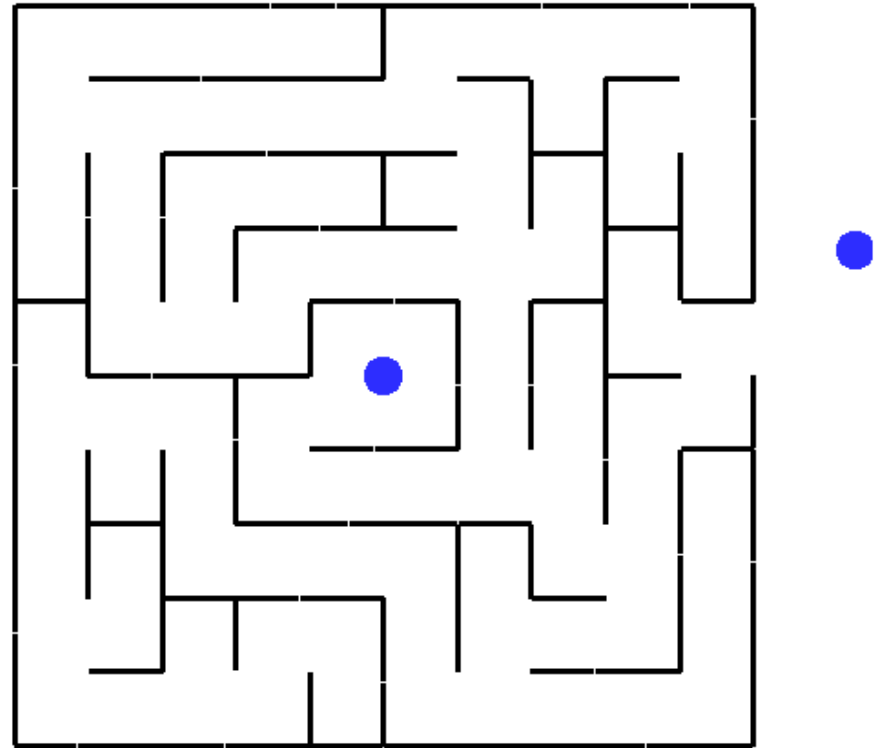
**Water-tight surfaces** are necessary for solid/fluid simulations  
How to detect leakages?



It is like solving a 3D labyrinth

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How to detect leakages?

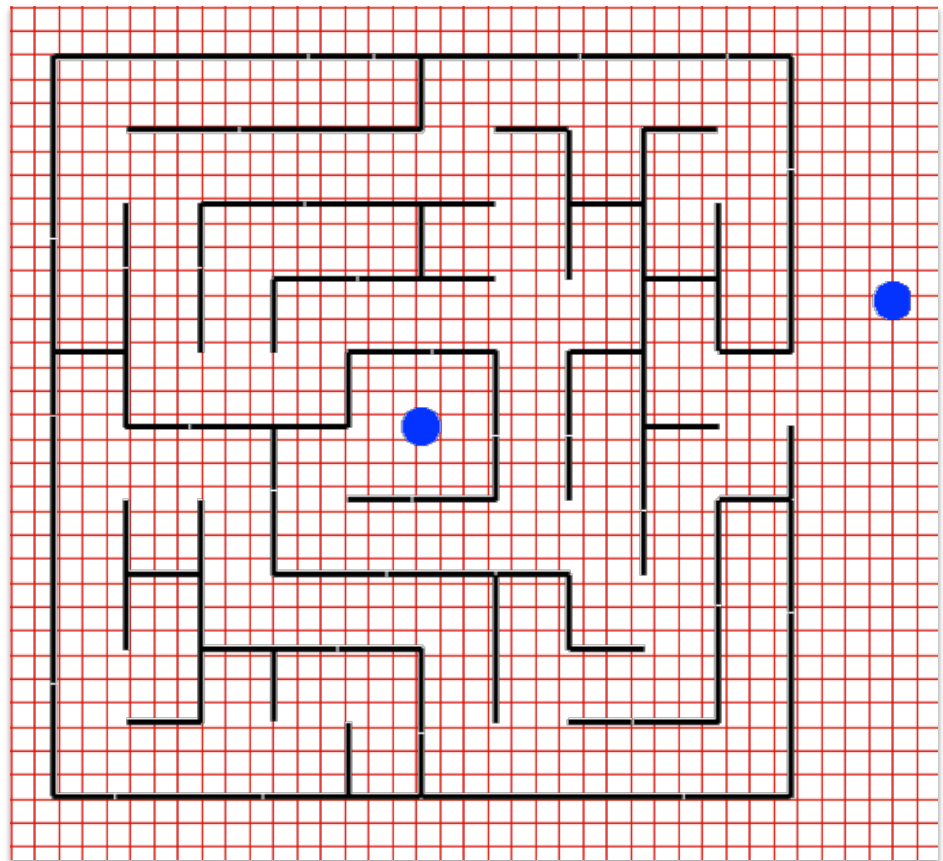


**It is like solving a 3D labyrinth**  
Find the path...

# Find Leakages

**Water-tight surfaces** are necessary for solid/fluid simulations  
How to detect leakages?

**Step 1:** create a volume grid and identify all the face intersection (ray tracing)



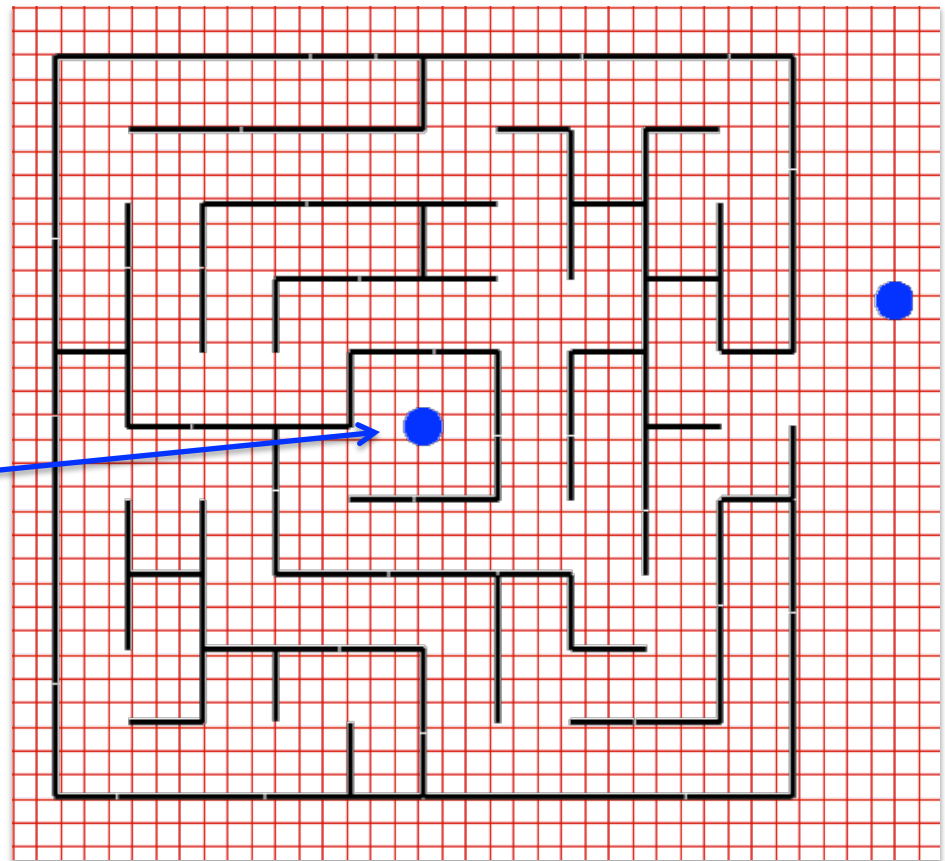
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**Step 2:** solve the eikonal equation with a front marching algorithm from the center

$$|\nabla u| = 1$$



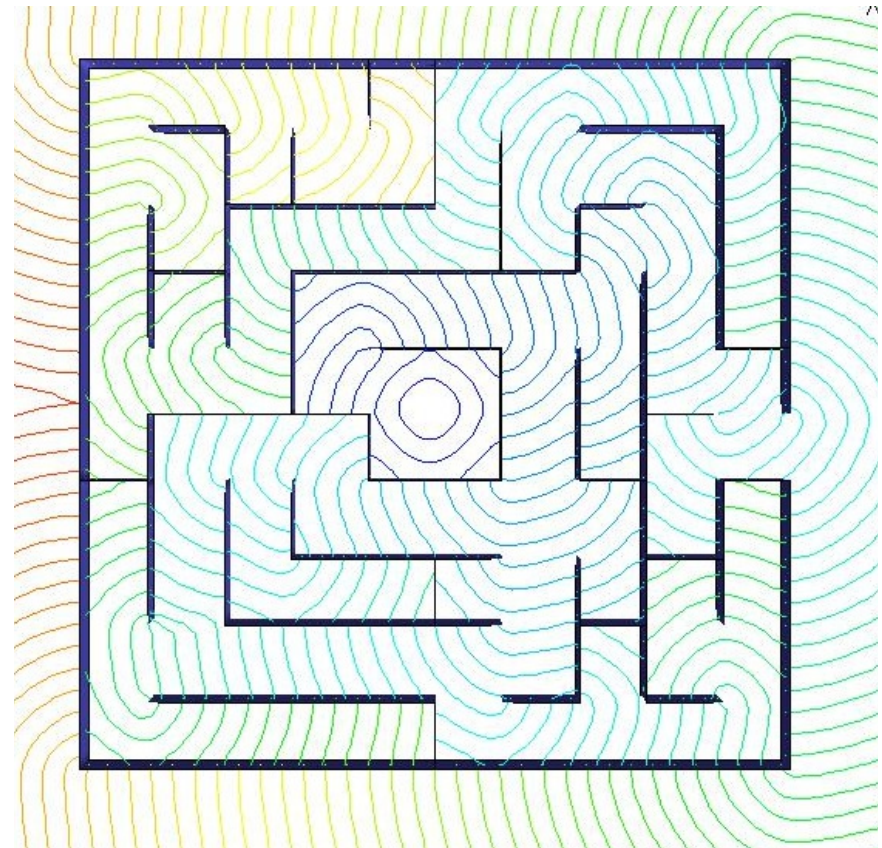
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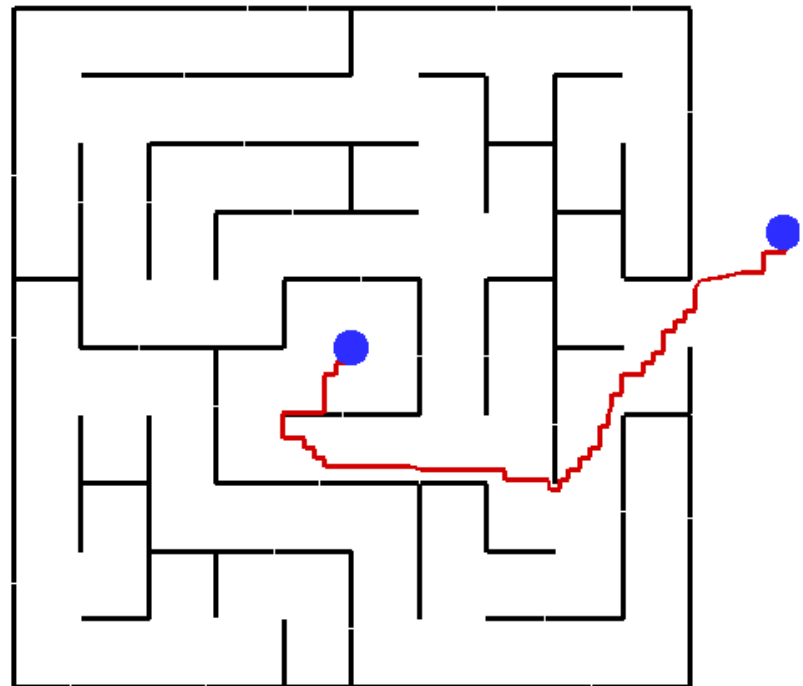
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How to detect leakages?

**Step 1:** create a volume grid and identify all the face intersection (ray tracing)

**Step 2:** solve the eikonal equation with a front marching algorithm from the center

**Step 3:** navigate the field and create a leakage path...



# An example: Find Leakages

For a full car assembly the algorithm is able to automatically detect a leakage path associated to the tire and traced back to the absence of valve stems and caps from the part database!

